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# An assessment of real-gas modelling in 2D enclosures

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## Abstract

In order to model efficiently the radiative transfer in a real-participating gas, various methods have been developed during the last few decades. Each method has its own formulation and leads to different accuracies and computation times. Most of the studies reported in the literature concern specific real-gas models, and very few are devoted to an extended comparison of these models. The present study is a 2D assessment of the main real-gas methods: the cumulative- $k$  method (CK), the statistical narrow-band model (SNB), the hybrid SNB-CK method, the grey-band method (GB), the weighted sum of grey gases method (WSGG), the spectral line-based weighted sum of grey gases method (SLW) and the exponential wide band model (EWB). Five cases have been considered: two homogeneous and isothermal cases with a single participating gas ( $\text{CO}_2$  and  $\text{H}_2\text{O}$ ), two non-homogeneous and non-isothermal cases with a single participating gas ( $\text{CO}_2$  and  $\text{H}_2\text{O}$ ), and one homogeneous and non-isothermal case with a mixture of  $\text{CO}_2$  and  $\text{H}_2\text{O}$ . Although the SNB and SNB-CK methods are the most accurate methods, the SLW method seems actually the best deal between accuracy and computation time. © 1999 Elsevier Science Ltd. All rights reserved.

**Keywords:** Non-grey gas models; Discrete ordinates method; Ray tracing

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## 1. Introduction

In the last three to four decades, the numerical evaluation of radiative transfer phenomena has drawn an increasing interest in the field of combustion, remote sensing and atmospheric media. The determination of radiative fluxes at surfaces and radiative volumetric terms in media containing real gases (principally  $\text{H}_2\text{O}$  and  $\text{CO}_2$  in the combustion products of hydrocarbon fuels) is difficult because of the strong dependence of the absorption coefficient of these gases on the wave

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## Nomenclature

$A_j(s \rightarrow s')$	the $j$ th band absorptance along a path between $s$ and $s'$
$a_i(T)$	weighting factor for the $i$ th grey gas of the WSGG method at temperature $T$
$a_i(s)$	weighting factor for the $i$ th cross section of the SLW method
$c$	mole fraction of radiating gas
$C_{\text{abs}}(s)$	absorption cross section at a given position $s$ ( $\text{m}^2/\text{mol}$ )
$(\text{div } \mathbf{q})_x$	flux divergence along $x$ ( $\text{W}/\text{m}^3$ )
$f(\kappa)$	distribution function of the absorption coefficient $\kappa$
$g(\kappa)$	cumulative distribution function of the absorption coefficient $\kappa$
$g_j$	value of the pseudo wave number for the $j$ th point of the Gauss–Lobatto quadrature
$I^\circ$	blackbody intensity ( $\text{W}/\text{m}^2 \text{ sr}$ )
$q_k$	surface flux at wall $k$ ( $\text{W}/\text{m}^2$ )
$l_m$	mean path length (m)
$N(s)$	molar density at a given position $s$ ( $\text{mol}/\text{m}^3$ )
$n$	number of Gauss–Lobatto quadrature points
$p$	pressure of the gas medium (atm)
$p_g$	partial pressure of participating gas (atm)
$s$	distance on the path of a ray (m)

## Greek symbols

$\Delta F$	fraction of blackbody radiation emitted in an interval $\Delta\nu$
$\Delta\nu$	wave-number interval ( $\text{cm}^{-1}$ )
$\kappa$	absorption coefficient ( $\text{m}^{-1}$ )
$K$	absorption coefficient ( $\text{m}^{-1} \text{ atm}^{-1}$ )
$\tilde{\omega}_j$	weighting factor associated with the $j$ th point of the Gaussian quadrature for the CK method
$\omega_m$	weighting factor associated with the $m$ th direction of the $T_N$ quadrature
$\Omega$	direction of radiation propagation

## Subscripts

$c$	stands for $\text{CO}_2$
$i, j$	a given point of the Gauss–Lobatto quadrature for the CK and SNB-CK methods, or a given cross section for the SLW method or a given grey gas for the WSGG method
$m$	a given value of the angular quadrature
$w$	stands for $\text{H}_2\text{O}$

## Greek subscript

$\Delta\nu$	over a wave-number interval ( $\text{cm}^{-1}$ )
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number. Nevertheless, various models have been developed, some only adapted to the computation of total quantities (integrated over the entire wave number spectrum, as in the case of combustion chambers), others also suited to low resolution quantities (e.g. for the infrared signature of rocket plumes). The accuracy of the results yielded by the latter, the so-called narrow-band methods, is better than that produced by the global methods, but these methods also require higher computation time. As an example, the line-by-line method, which necessitates some  $10^6$  resolutions of the RTE, is often used as a reference method for the validation of new techniques, but is not economically applicable to engineering problems.

Two of the most widely used band methods over the last ten years are the statistical narrow-band (SNB) and the correlated  $k$  (CK) distribution models. On the one hand, studies have been conducted to establish new databases [1] for these two methods with the help of spectroscopic analyses that take into account the emission lines at high temperatures [2]. On the other hand, the coupling of these models with the discrete ordinates method (DOM) has also been investigated. The discrete ordinates method is indeed very popular among the researchers, owing to its accuracy, its low computation time and its formulation in absorption coefficients. For these reasons, most of the couplings have been done with different versions of DOM. However, in some cases, the discrete transfer method and the ray-tracing method have also been used [3]. When performing the coupling of SNB and DOM in one dimension, attention has been given to the decorrelations created by the reflections at non-black walls of an enclosure. In an attempt to account for this phenomenon, Kim et al. [4–6] suggested a modification of the boundary conditions. In another study, de Miranda and Sacadura [7,8] proposed an uncorrelated formulation of the RTE in one dimension, which minimizes significantly the computation time, but to the detriment of accuracy. However, their uncorrelated formulation is still based on the gas transmissivity, which makes it difficult to solve in multidimensional geometries since solvers based on gas absorption coefficient such as the popular DOM cannot be used. It was demonstrated by Liu et al. [9] that the uncorrelated expression of de Miranda and Sacadura can be formulated in terms of the gas absorption coefficient derived from the mean pathlength of the local computational cell. Their formulation is termed ‘grey-band’ model (GB) since in this model the real gas is treated as grey in each narrow band. This procedure makes the extension of the uncorrelated formulation to multidimensional enclosures straightforward.

The CK model was initially developed for atmospheric modelling applications [10–12]. It has the advantage over the SNB model that scattering effects can be accounted for (this is not possible with the SNB model, unless the Monte Carlo method is used in conjunction, which increases considerably the computation time). This method has been recently evaluated by Pierrot [13,14] against other currently used methods for one-dimensional cases at high temperatures (up to 2500 K). For situations of high-temperature gradients, a modified technique (the CKFG method) is rather suggested [15,16].

The third popular band method is the exponential wide band (EWB), first proposed by Edwards and Balakhrisan [17]. This model is less demanding in computation time than the SNB and is thus used in engineering problems when simultaneous mathematical modelling of combustion, radiation and flow pattern is sought [18–21].

As for the global methods, the basic concept is that of the weighted sum of grey gases (WSGG) method, initially developed by Hottel and Sarofim [22]. The most widely used classical WSGG model parameters are those of Smith et al. [23] and Farag and Allam [24,25]. Recent ones by

Soufiani et al. [14,26] have also been worked out. The general concept of this method was recently adapted by Denison and Webb [27–30], who developed the spectral line based weighted sum of grey gases (SLW) model. These authors obtained very good results (good accuracy and low computation time) for one-dimensional problems in non-scattering media. More recently, Kim and Song [31] have applied the WSGG concept to develop narrow-band models for the computation of low-resolution spectral intensities.

Whereas a number of studies have been conducted on the radiative behaviour of real gases in one-dimensional geometry (infinite slab) in order to solve specific problems occurring with the individual methods or their coupling with the RTE, very few published analyses are devoted to either cases of higher dimensions [18,19] or thorough comparisons of different real-gas models [14].

In this paper, the accuracy and efficiency of the most generally used real-gas models are compared in two-dimensional geometries. The real-gas models chosen for comparison purposes are the CK, SNB, hybrid SNB-CK, Grey-Band, WSGG, SLW and EWB methods. These models are coupled either with a ray-tracing technique or two different DOM procedures (one of them is due to Sakami et al. [32] and the other is that of Carlson and Lathrop [33]). Two cases are reported with CO<sub>2</sub> as the only participating constituent: one homogeneous and isothermal, the other non-homogeneous and non-isothermal. Two similar cases are studied with H<sub>2</sub>O as the only radiating gas. Finally, a fifth case considers a non-isothermal and homogeneous mixture of CO<sub>2</sub> and H<sub>2</sub>O.

## 2. The RTE solvers used

As a result of the strong interest created by the discrete ordinates method, different versions of the method (e.g. Heart/diamond/step/exponential scheme, El Wakil method [34]) were developed in parallel by a number of researchers. Consequently, since the work reported in the present paper is a joint contribution from two separate research bodies, two DOM versions were used. One of these is due to Sakami et al. [32]. This version is specially adapted to radiative transfer in enclosures of complex geometry. Since the grid used is triangular (2D), the usual interpolations (forward or central differencing) are difficult to apply and this led to the implementation of a closure procedure that consists in solving rigorously the RTE monodimensionally in each angular direction  $\bar{\Omega}_m$  over the entire domain bounded by the grid cells. The other is the conventional DOM due to Carlson and Lathrop [33]; the positive differencing scheme is used in the present calculations.

For the SNB model, formulated in mean transmissivity, the RTE solver used is a ray-tracing method. The domain of the problem is first divided into a number of control cells where the physical properties of the gas are assumed to be uniform. Then, the calculation of radiation intensity at a spatial point and for a given angular direction proceeds in two steps. The first step consists in determining the starting point of the ray. From the given spatial point, the angular direction is followed backwards until it intersects a bounding surface. Then, from this intersection point on the wall, the radiation intensity along the line-of-sight is calculated in the forward direction using the discretised form of the band average RTE, until the point of interest is reached. An analogous procedure is repeated for all the control surface centres and for all the directions of

the quadrature. The radiative volumetric term at the centre of each control cell is calculated from the divergence of the heat flux.

### 3. The real-gas models

The comparative behaviour of the following models will be analysed in this paper: CK, SNB, hybrid CK-SNB, grey-band, WSGG, SLW and EWB. A detailed description of these models can be found in the literature. However, in order to recall their main features, a brief description of each model is given below.

#### 3.1. The CK model

The CK method consists in subdividing the entire spectrum into narrow-bands of width  $\Delta\nu$  and then the radiative transfer over each of them is solved. To integrate the radiative quantities over the narrow bands, two variables are defined. The first one is the distribution function  $f_{\Delta\nu}(\kappa)$  which is defined in such a way that  $f_{\Delta\nu}(\kappa)d\kappa$  represents the fraction of  $\Delta\nu$  where the absorption coefficient lies between  $\kappa$  and  $\kappa + d\kappa$ . Then, the cumulative distribution function  $g_{\Delta\nu}(\kappa)$  of the absorption coefficient is introduced; it represents the probability that the absorption coefficient is less than  $\kappa$  in the bandwidth  $\Delta\nu$  and can be interpreted as a pseudo wave number varying between 0 and 1 [12,16].

When using this variable and introducing the integrated value of the blackbody intensity over  $\Delta\nu$ ,  $\overline{\Delta I}_{\Delta\nu}^o = \Delta F \sigma T^4 / \pi$  [19], the integrated RTE over each band becomes

$$\int_0^1 \frac{\partial \overline{\Delta I}_{\Delta\nu,m,g}}{\partial s} dg = - \int_0^1 \kappa_{\Delta\nu}(g) [\overline{\Delta I}_{\Delta\nu,m,g}(s) - \overline{\Delta I}_{\Delta\nu}^o(s)] dg. \quad (1)$$

Since  $g_{\Delta\nu}(\kappa)$  is a monotonically increasing (bijective) function, the integration over the  $g$  space can be conducted by using a Gauss–Lobatto quadrature. Therefore, as a first step, the RTE is solved for each quadrature point  $g_i$ :

$$\frac{\partial \overline{\Delta I}_{\Delta\nu,m,i}}{\partial s} = - \kappa_{\Delta\nu}(g_i) [\overline{\Delta I}_{\Delta\nu,m,i}(s) - \overline{\Delta I}_{\Delta\nu}^o(s)]. \quad (2)$$

The total intensities are obtained in the following manner:

$$\overline{\Delta I}_{\Delta\nu,m}(s) = \sum_{i=1}^n \tilde{\omega}_i \overline{\Delta I}_{\Delta\nu,m,i}(s) \quad \text{and} \quad I(s) = \sum_{\Delta\nu} \sum_m \omega_m \overline{\Delta I}_{\Delta\nu,m}(s), \quad (3)$$

where the  $\tilde{\omega}_i$ 's are the corresponding weighting functions of the Gauss–Lobatto quadrature,  $n$  is the number of quadrature points considered and  $\omega_m$  is the weighting factor associated with the  $m$ th angular direction.

The absorption coefficients  $\kappa_{\Delta\nu}(g_i)$  used in this paper are those obtained by Taine et al. [1]. These were fitted to a seven-point quadrature with bandwidths varying from 100 to 400  $\text{cm}^{-1}$  (optimized

wide band version, for shorter computation time, of their initial method covering narrow-bandwidths of  $25 \text{ cm}^{-1}$ ). The number of optimized CK bands for  $\text{CO}_2$  and  $\text{H}_2\text{O}$  is 21 and 43, respectively, instead of 367 for the narrow-bands version for each of the two gases.

If the medium is a mixture of two radiating gases, say  $\text{CO}_2$  and  $\text{H}_2\text{O}$ , the problem becomes more complicated at the overlapping bands. The formulation of the RTE becomes

$$\frac{\partial \overline{\Delta I}_{\Delta v, m, i, j}(s)}{\partial s} = -[\kappa_{\Delta v, c}(g_i) + \kappa_{\Delta v, w}(g_j)](\overline{\Delta I}_{\Delta v, m, i, j}(s) - \overline{\Delta I}_{\Delta v}^0(s)). \quad (4)$$

This comes from the fact that the mean transmissivity of a mixture is obtained by using the mean transmissivity of its components as follows:

$$\tau = \tau_c \tau_w = \left( \sum_{i=1}^7 \tilde{\omega}_i \exp(-\kappa_{\Delta v, c}(g_i)l) \right) \left( \sum_{j=1}^7 \tilde{\omega}_j \exp(-\kappa_{\Delta v, w}(g_j)l) \right), \quad (5)$$

which can be re-written as

$$\tau = \sum_{i=1}^7 \sum_{j=1}^7 \tilde{\omega}_i \tilde{\omega}_j \exp[-(\kappa_{\Delta v, c}(g_i) + \kappa_{\Delta v, w}(g_j))l]. \quad (6)$$

Finally, the total intensity is obtained by

$$I(s) = \sum_{\Delta v} \sum_m \omega_m \sum_{i=1}^7 \tilde{\omega}_i \sum_{j=1}^7 \tilde{\omega}_j \overline{\Delta I}_{\Delta v, m, i, j}(s). \quad (7)$$

### 3.2. The SNB model

Similarly to the CK method, the SNB model consists in subdividing the spectrum into small wave number intervals  $\Delta v$ , and then calculating the radiative transfer on each of these narrow-bands. For this purpose, a mean band transmissivity is defined as

$$\bar{\tau}_v(s' \rightarrow s) = \exp \left[ -\frac{\bar{\beta}_v}{\pi} \left( \sqrt{1 + \frac{2\pi c p |s' \rightarrow s| \bar{k}_v}{\bar{\beta}_v}} - 1 \right) \right], \quad (8)$$

where the average bandwidth to spacing ratio is given by  $\bar{\beta}_v = 2\pi(\bar{\gamma}_v/\bar{\delta}_v)$ . A number of distribution laws are available for determining the various parameters of this equation [15,16]. However, for a non-isothermal and non-homogeneous path, the Curtis–Godson approximation is recommended [11,15,16]. The updated SNB model parameters provided by Soufiani [1] were used in these calculations. This data set contains SNB model parameters for  $\text{CO}$ ,  $\text{CO}_2$  and  $\text{H}_2\text{O}$  with a constant spectral interval of  $25 \text{ cm}^{-1}$  width. The covered temperature and spectral ranges are 300–2900 K and 150–9300  $\text{cm}^{-1}$ , respectively.

The discretized form of the narrow-band averaged RTE along a line-of-sight for high emissivity surrounding walls is given as [4]

$$\bar{I}_{\Delta v, m}(i+1) = \bar{I}_{\Delta v, m}(i) + [1 - \bar{\tau}_{\Delta v, m}(i \rightarrow i+1)] \bar{I}_{\Delta v}^0(i+1/2) + \bar{C}_{\Delta v, m}(i+1/2) \quad (9)$$

with

$$\begin{aligned}\bar{C}_{\Delta v, m}(i + 1/2) = & I_{w\Delta v, m}(1)[\bar{\tau}_{\Delta v, m}(1 \rightarrow i + 1) - \bar{\tau}_{\Delta v, m}(1 \rightarrow i)] \\ & + \sum_{k=1}^{i-1} [(\bar{\tau}_{\Delta v, m}(k + 1 \rightarrow i + 1) - \bar{\tau}_{\Delta v, m}(k + 1 \rightarrow i)) \\ & - (\bar{\tau}_{\Delta v, m}(k \rightarrow i + 1) - \bar{\tau}_{\Delta v, m}(k \rightarrow i))]I_{\Delta v}^o(k + 1/2),\end{aligned}\quad (10)$$

which arises from the correlation between the spectral gas absorption coefficient and the spectral radiation intensity. This term, along with the ray-tracing nature of the RTE solution, dramatically increases the computation time of the SNB model. The spatial discretization index  $i = 1$  corresponds to the starting point of the line-of-sight on a wall boundary.

The total intensity at a given point is calculated as follows:

$$I(s) = \sum_{\Delta v} \sum_m \bar{I}_{\Delta v, m}(s) \Delta v. \quad (11)$$

For a gas mixture of  $H_2O$  and  $CO_2$ , the gas transmissivity at an overlapping band is obtained by multiplying the transmissivity of each component [see Eq. (6)].

$$\tau_{\text{mixt}} = \tau_c \times \tau_w. \quad (12)$$

### 3.3. The hybrid SNB-CK model

This method is essentially the CK approach described above. The difference lies in how the cumulative distribution function  $g_{\Delta v}(\kappa)$  is calculated. In the hybrid SNB-CK model, the distribution function  $f(\kappa)$  is first obtained by inverse Laplace transformation of the SNB gas transmissivity,

$$\tau_v(L) = 1/\Delta v \int_{\Delta v} \exp(-\kappa_v L) dv = \int_0^\infty f(\kappa) \exp(-\kappa L) d\kappa.$$

As demonstrated by Domoto [35], the analytical expression of  $f(\kappa)$  for a Malkmus model is

$$f(\kappa) = \frac{1}{2} \kappa^{-3/2} (BS)^{1/2} \exp\left(\frac{\pi B}{4} \left(2 - \frac{S}{\kappa} - \frac{\kappa}{S}\right)\right). \quad (13)$$

Then the cumulative distribution function  $g_{\Delta v}(\kappa)$  can be obtained analytically. Its expression has been derived by Lacis and Oinas [12] as

$$g_{\Delta v}(\kappa) = \frac{1}{2} \left[ 1 - \operatorname{erf}\left(\frac{a}{\sqrt{\kappa}} - b\sqrt{\kappa}\right) \right] + \frac{1}{2} \left[ 1 - \operatorname{erf}\left(\frac{a}{\sqrt{\kappa}} + b\sqrt{\kappa}\right) \right] \exp(\pi B), \quad (14)$$

where  $a = \sqrt{\pi BS}/2$ ,  $b = \sqrt{\pi B/S}/2$ ,  $B = 2\bar{\beta}/\pi^2$ ,  $S = \bar{k}fp$ , and  $\operatorname{erf}(x)$  is the error function defined as

$$\operatorname{erf}(x) = \frac{2}{\sqrt{\pi}} \int_0^x \exp(-t^2) dt. \quad (15)$$

For each Gauss–Lobatto quadrature point  $g_i$ , the corresponding gas absorption coefficient  $k_i = k(g_i)_{\Delta v}$  is calculated by inverting the cumulative distribution function given by Eq. (14). A Newton–Raphson-type iteration method is used to perform the inversion. The iteration converges very rapidly, requiring only 5–10 iterations to achieve 6-digit precision, if the inflection point at  $\kappa = \kappa_{\max}$ , where the distribution function of the absorption coefficient  $f(\kappa)$  peaks, is taken as the starting point for the iteration, as suggested by Lacis and Oinas [12]. The SNB model parameters used to perform the hybrid SNB-CK calculations are the same as those reported in Section 3.2. The RTE associated with the hybrid SNB-CK method is identical to that used with the CK method.

For a mixture, the procedure described above for a single radiating gas is applied to each component to obtain  $\kappa_c(g_i)_{\Delta v}$  and  $\kappa_w(g_j)_{\Delta v}$ :

$$\begin{aligned} \sum_{i=1}^7 \tilde{\omega}_i \exp[-\kappa_c(g_i)_{\Delta v} L] &= \tau_{c,SNB}, \\ \sum_{j=1}^7 \tilde{\omega}_j \exp[-\kappa_w(g_j)_{\Delta v} L] &= \tau_{w,SNB}. \end{aligned} \quad (16)$$

Then the total intensities are obtained by Eqs. (4) and (7), similarly to the CK method.

### 3.4. The grey-band model

This model assumes that the radiating gases behave like a grey gas in each narrow-band. The following form of the RTE is solved

$$\frac{\partial \bar{I}_{\Delta v,m}(s)}{\partial s} = -\bar{\kappa}_{\Delta v}(s)[\bar{I}_{\Delta v,m}(s) - \bar{I}_{\Delta v}^o(s)]. \quad (17)$$

The grey narrow-band gas absorption coefficient is estimated by:  $\bar{\kappa}_{\Delta v} = -(\ln \bar{\tau}_{\Delta v}(l_m))/l_m$ , where  $l_m = 3.6V/S$  is the mean path-length of the local computational control volume [22]. The total intensity  $I(s)$  is obtained from:

$$I(s) = \sum_{\Delta v} \sum_m \bar{I}_{\Delta v,m}(s) \Delta v. \quad (18)$$

### 3.5. The WSGG model

This method considers a real gas as a sum of a number of grey gases (generally 3–5). It was developed by Hottel and Sarofim as a means to study media of non-uniform temperature. Whereas the absorption coefficients of the different grey gases are fixed throughout the enclosure, the weighting factors are strongly dependent on the local temperatures. The total emissivity is expressed as  $\varepsilon(T) = \sum_{i=0}^n a_i(T)(1 - e^{-K_i p \varepsilon})$ , and the RTE takes the form [36]:

$$\frac{dI_{m,i}(s)}{ds} = \kappa_i[(a_i(s)I^o(s)) - I_{m,i}(s)] \quad (19)$$

with  $\kappa_i = K_i \times p_g$  ( $p_g$  is the partial pressure of the participating gas). The total intensity is given by

$$I(s) = \sum_i \sum_m \omega_m I_{m,i}(s). \quad (20)$$

The weighting factors can be found in Smith et al. [23], Farag and Allam [24,25] or Soufiani et al. [14,26].

### 3.6. The SLW model

The WSGG model is well adapted to non-isothermal media, however, it cannot handle non-homogeneous media. To circumvent this limitation, Denison and Webb [27–30] developed the SLW method. The procedure consists in choosing a number of absorption cross-sections and calculating the radiative transfer occurring between two consecutive cross-sections. The associated weighting factors depend not only on the local temperatures (as in the WSGG) but also on the local concentrations. Reference values, which are taken as the mean temperature and concentration over the domain, constitute important parameters in this method, and this can also have a non-negligible effect on the weighting factors. The choice of these reference values plays an important role in determining the spectral subdivisions related to each absorption section.

The form of the RTE that is to be solved is analogous to that of the WSGG model. Between two successive absorption cross-sections  $C_{\text{abs},i}$  and  $C_{\text{abs},i+1}$  the RTE is written as

$$\frac{dI_{m,i}(s)}{ds} = N(s)C_{\text{abs},i}(s)[(a_i(s)I^0(s)) - I_{m,i}(s)], \quad (21)$$

where  $N(s)$  is the molar density at position  $s$ . The total intensity is obtained from

$$I(s) = \sum_m \sum_i \omega_m I_{m,i}(s). \quad (22)$$

In this paper we used 15 cross-sections logarithmically spaced between  $3 \times 10^{-5}$  and  $600 \text{ m}^2/\text{mol}$  for  $\text{CO}_2$  and between  $3 \times 10^{-5}$  and  $60 \text{ m}^2/\text{mol}$  for  $\text{H}_2\text{O}$  [30].

For a mixture of  $\text{H}_2\text{O}$  and  $\text{CO}_2$ , the combinations between all the absorption cross-sections of the two species have to be considered. The RTE to be solved becomes:

$$\frac{dI_{m,i,j}(s)}{ds} = \kappa_{i,j}(s)[(a_{i,j}(s)I^0(s)) - I_{m,i,j}(s)], \quad (23)$$

where  $\kappa_{i,j}(s) = N_c C_{\text{abs},c,i}(s) + N_w C_{\text{abs},w,j}(s)$ , and  $a_{i,j}(s) = a_{c,i}(s) \times a_{w,j}(s)$ . The total intensity is given by

$$I(s) = \sum_m \sum_i \sum_j \omega_m I_{m,i,j}(s). \quad (24)$$

### 3.7. The EWB model

Following Kim et al. [4], the discretized transfer equation associated with a wide band model can be written as

$$I_{i+1} = I_i + \sum_{j=1}^J I_{b,j,i+1/2} A_j(s_i \rightarrow s_{i+1}) - \sum_{j=1}^J \bar{I}_{w,j} [A_j(s_w \rightarrow s_{i+1}) - A_j(s_w \rightarrow s_i)] - \sum_{j=1}^J \sum_{k=1}^{i-1} \bar{I}_{b,j,k+1/2} [A_j(s_{k+1} \rightarrow s_{i+1}) - A_j(s_k \rightarrow s_{i+1}) - A_j(s_{k+1} \rightarrow s_i) + A_j(s_k \rightarrow s_i)], \quad (25)$$

where  $I$  is the integrated radiation intensity over the entire wave number spectrum and  $A_j(s_i \rightarrow s_{i+1})$  is the  $j$ th band absorptance along a path between  $s_i$  and  $s_{i+1}$ . The exponential wide band (EWB) model is used to obtain the band absorptance. This absorptance is determined by three band parameters: the integrated band intensity  $\alpha$ , the exponential decay width  $\omega$ , and the mean line-width-to-spacing parameter  $\beta$ . Formulations of these three parameters are given in Edwards [37] and in Lallemand and Weber [21]. These are dependent on the fundamental band parameters such as their spectral locations, their values at the reference temperature, the number of modes of vibration, and the statistical weighting factor for degeneracy. These fundamental parameters for CO<sub>2</sub> and H<sub>2</sub>O summarized by Edwards [37] are used in the present EWB model calculations, together with the procedure for efficient computation proposed by Lallemand and Weber [21]. An exception is made, however, for the pure rotational band of H<sub>2</sub>O, where the parameters recommended by Modak [38] are employed. The wide band absorptance  $A_j$  is then calculated using the method described in detail by Edwards [37]. For a non-isothermal and/or non-homogeneous path, the recommendations of Edwards are also used for estimating the equivalent model parameters. The EWB model calculations are performed only for a single participating gas, either CO<sub>2</sub> or H<sub>2</sub>O.

The above transfer equation for the EWB model is then solved using the ray-tracing method. Similarly to the hybrid SNB-CK approach, the hybrid EWB-CK method can also be followed to

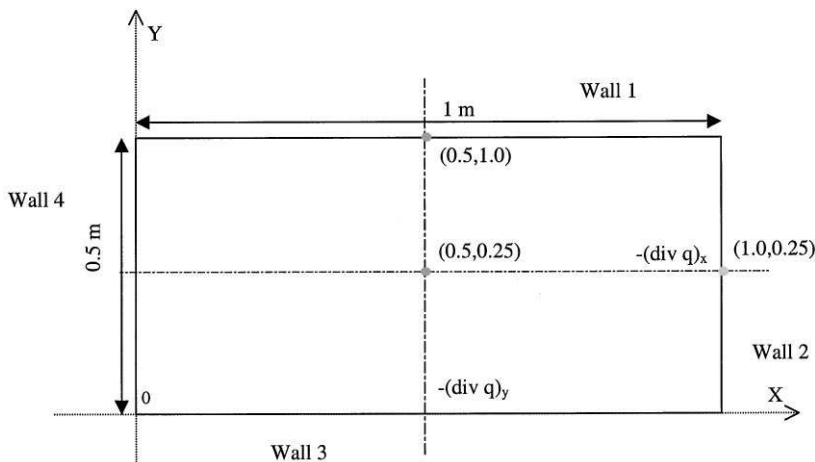


Fig. 1. The enclosure.

implement the EWB model into the RTE as demonstrated by Marin and Buckius [39]. The EWB-CK method is not pursued in this study.

#### 4. Description of the tests

The enclosure used for all the tests is rectangular ( $1 \times 0.5$  m) and its walls are black and kept at 0 K (Fig. 1). The grid is uniform ( $61 \times 31$ ) and the  $T_7$  quadrature due to Thurgood [40] is adopted. In order to make meaningful comparisons, we have chosen five different kinds of participating media: two homogeneous and isothermal cases with a single participating gas ( $\text{CO}_2$  and  $\text{H}_2\text{O}$ ), two non-homogeneous and non-isothermal cases with a single participating gas ( $\text{CO}_2$  and  $\text{H}_2\text{O}$ ), and one homogeneous and non-isothermal case with a mixture of  $\text{CO}_2$

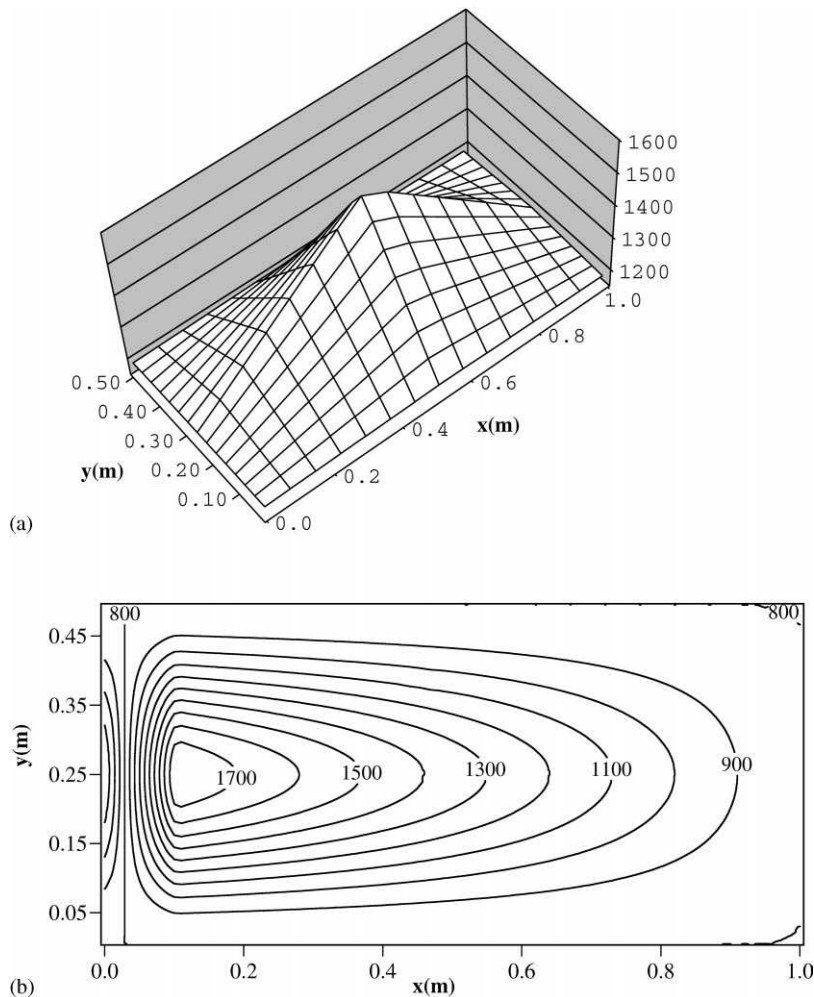


Fig. 2. (a) The prescribed non-isothermal temperature field (Cases 2 and 4); (b) The prescribed flame temperature,

Table 1  
Test conditions

Case	Participating gas	Temperature (K)	Concentration
1	CO <sub>2</sub>	Isothermal: 1000	Homogeneous: 10%
2	CO <sub>2</sub>	Non-isothermal: Eq. (26)	Non-homogeneous: Eq. (26)
3	H <sub>2</sub> O	Isothermal: 1000	Homogeneous: 20%
4	H <sub>2</sub> O	Non-isothermal: Eq. (26)	Non-homogeneous: Eq. (26)
5	CO <sub>2</sub> + H <sub>2</sub> O	Non- isothermal: Eq. (27)	Homogeneous: 10% CO <sub>2</sub> + 20% H <sub>2</sub> O

and H<sub>2</sub>O. For all the five cases, the non-participating gas is N<sub>2</sub> and the pressure of the gas medium is 1 atm. For the two isothermal cases, the enclosure is filled with, respectively, 10% CO<sub>2</sub> (Case 1) and 20% H<sub>2</sub>O (Case 3) at 1000 K. For the two non-isothermal cases with a single participating gas, the temperature and concentration (mole fraction) fields of the radiating gas are set as follows:

$$\begin{aligned} T(x, y) &= T_0[0.3333(1 - 2|x - 0.5|)(1 - 4|y - 0.25|) + 1], \\ c(x, y) &= c_0[4(1 - 2|x - 0.5|)(1 - 4|y - 0.25|) + 1], \end{aligned} \quad (26)$$

where  $T_0$  is 1200 K and  $c_0$  is 0.02 for CO<sub>2</sub> (case 2) and 0.04 for H<sub>2</sub>O (case 4). Hence, the gas temperature varies between 1200 and 1600 K (Fig. 2(a)) and the concentration field between 0.02 and 0.10 for case 2 and between 0.04 and 0.20 for case 4.

The fifth case is more representative of gas combustion (flame) in a furnace. The concentrations of the participating gases are assumed to be uniform in the enclosure: 10% CO<sub>2</sub> and 20% H<sub>2</sub>O; the temperature field is defined by the following expressions:

$$\begin{aligned} \text{for } x \leq 0.1, \quad T(x, y) &= (14000x - 400)(1 - 3y_0^2 + 2y_0^3) + 800, \\ \text{for } x \geq 0.1, \quad T(x, y) &= -\frac{10000}{9}(x - 1)(1 - 3y_0^2 + 2y_0^3) + 800 \end{aligned} \quad (27)$$

with  $y_0 = |0.25 - y|/0.25$  (Fig. 2(b)).

Table 1 summarizes the five different test conditions.

## 5. Results and discussion

### 5.1. Compatibility assessment of the RTE solvers

Before studying the real-gas cases, a test is made with a grey gas to ensure that the different RTE solvers will not influence the results obtained with the real-gas models. The value of the grey gas absorption coefficient is taken as  $0.5 \text{ m}^{-1}$ . The temperature of the medium is kept uniform at 1000 K and the walls are black and cold. Fig. 3 shows that the results yielded by the three RTE solvers are in good agreement: the maximum discrepancy is less than 0.7% for the wall heat fluxes and 0.6% for the heat flux divergences.

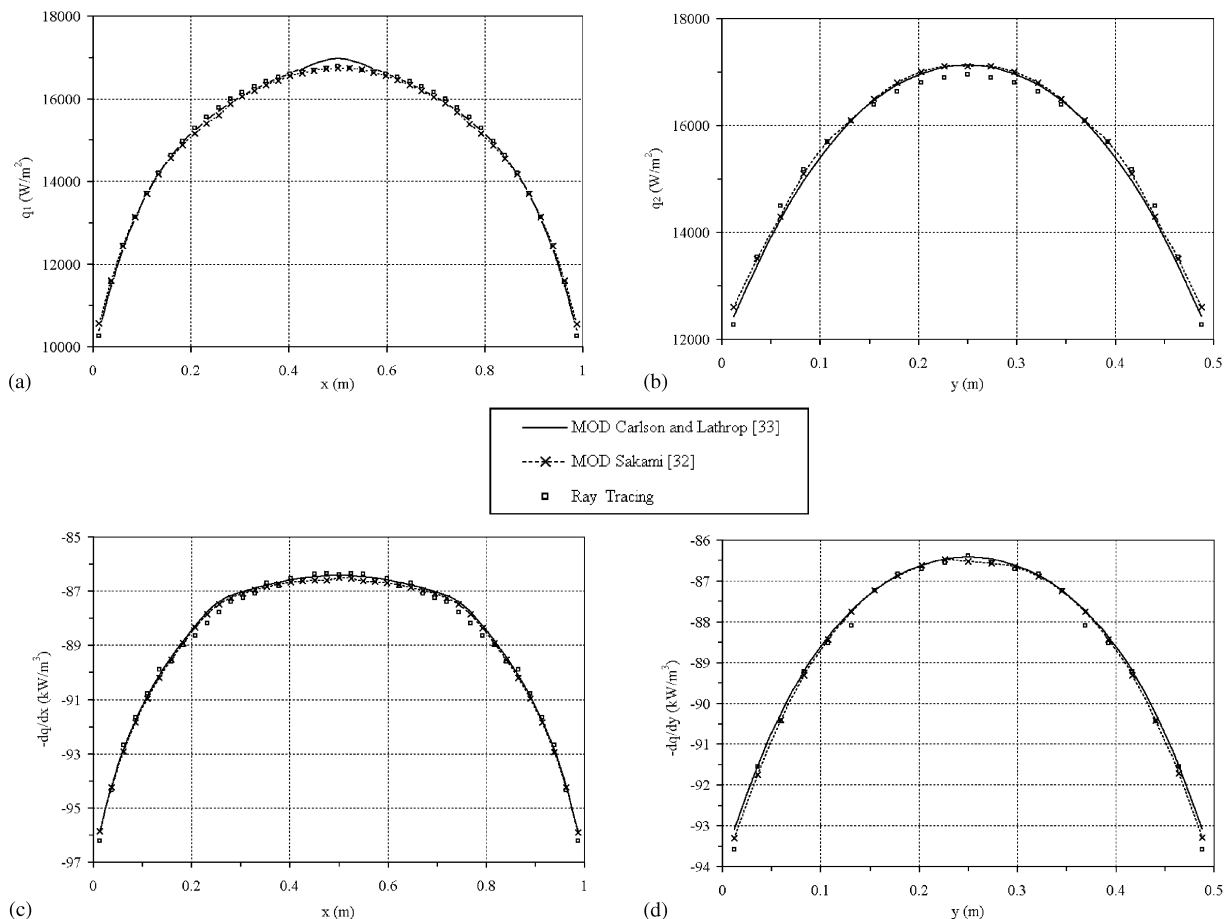
**Gray case**

Fig. 3. (a) Evolution of the heat flux  $q_1$  for the grey case; (b) Evolution of the heat flux  $q_2$  for the grey case; (c) Evolution of the source term  $-\text{div } q_x$  for the grey case; (d) Evolution of the source term  $-\text{div } q_y$  for the grey case.

## 5.2. Comparison of the real gas models

As can be observed in Figs. 4–8 presented hereafter, all the methods — except the grey gas model, which is given only as complementary information and is totally unreliable for the heat flux divergences — lead to similar distributions for the wall heat fluxes and the heat flux divergences. Therefore, before comparing quantitatively the results obtained with each individual method, a brief explanation of the general distributions is given below.

### 5.2.1. General distributions

For the two isothermal cases, the variation of the radiative source term, Figs. 4(c), (d), 6(c), and (d), is analogous to that obtained for similar one-dimensional cases with a real gas [6, 13, 30]: its

value is relatively small for the major portion of the enclosure and reaches large negative values very abruptly at the boundaries due to the cold walls. The variation of the heat flux is explained similarly: the lateral walls, vertical or horizontal depending on the displayed results, influence more their neighbouring cells than the middle ones, Figs. 4(a), (b), 6(a), and (b).

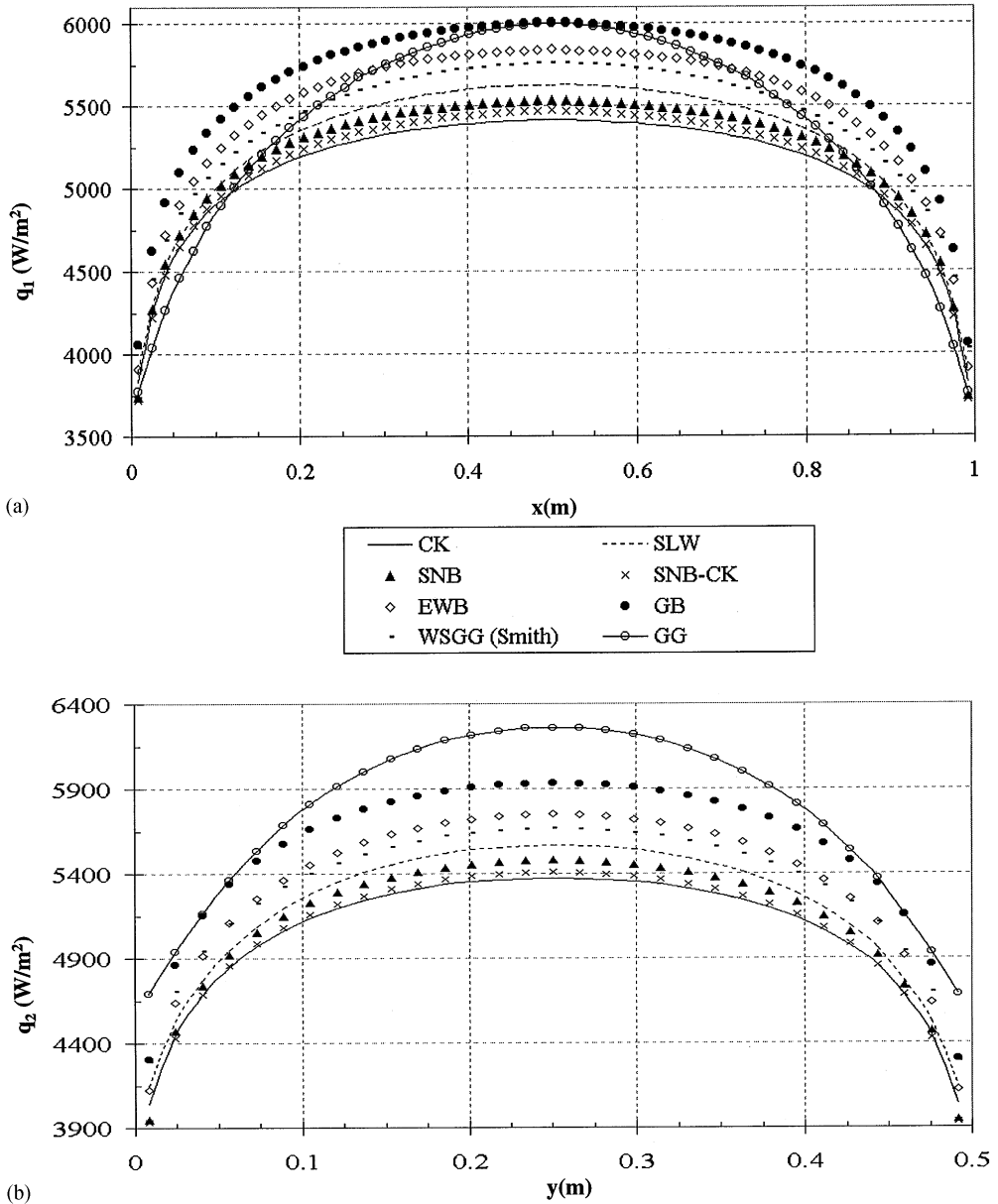


Fig. 4. (a) Evolution of the heat flux  $q_1$  for the isothermal and homogeneous case with CO<sub>2</sub> (Case 1); (b) Evolution of the heat flux  $q_2$  for the isothermal and homogeneous case with CO<sub>2</sub> (Case 1); (c) Evolution of the source term  $-(\text{div } q)_x$  for the isothermal and homogeneous case with CO<sub>2</sub> (Case 1); (d) Evolution of the source term  $-(\text{div } q)_y$  for the isothermal and homogeneous case with CO<sub>2</sub> (Case 1).

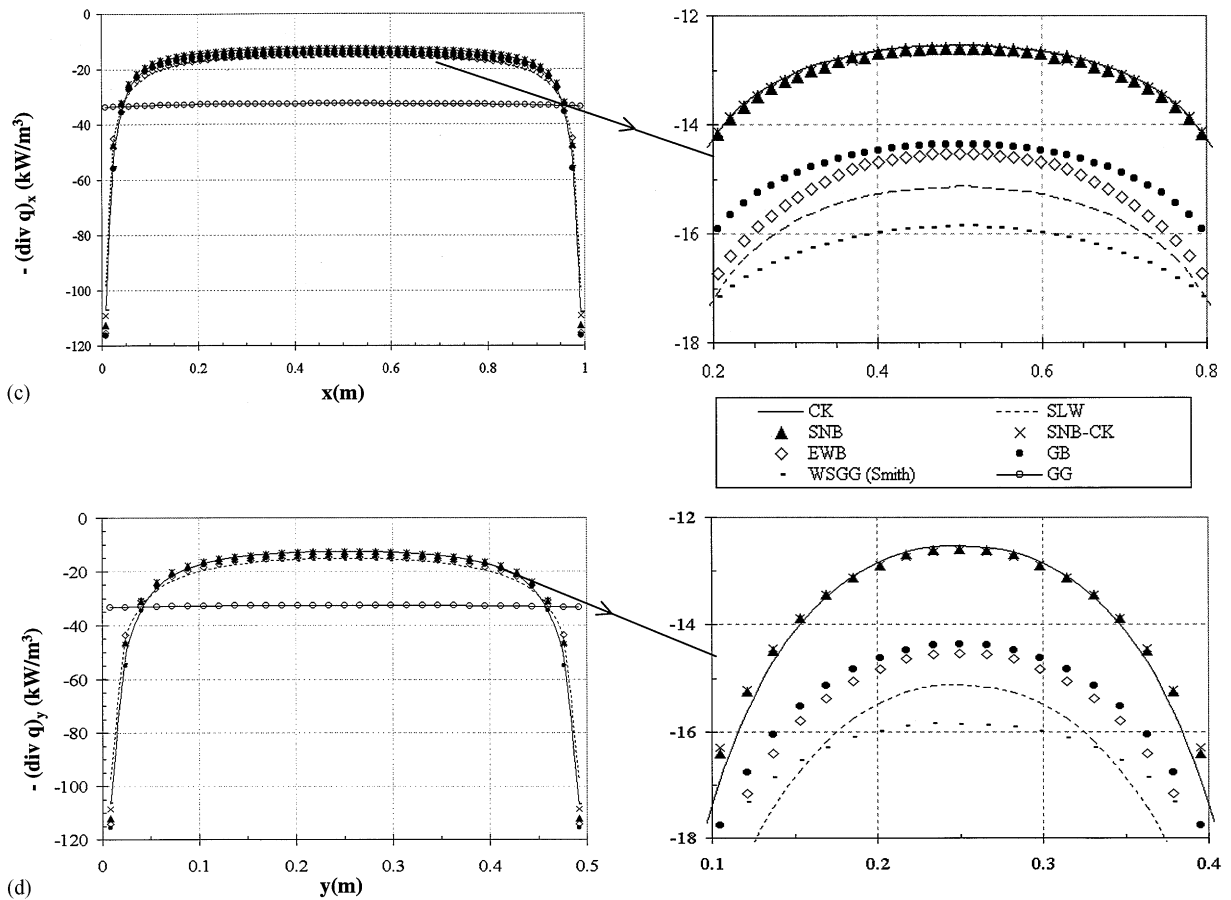


Fig. 4. (Continued).

For the non-homogeneous and non-isothermal cases (Cases 2 and 4), the variation of the divergence is quite different, Figs. 5(c),(d), 7(c), and (d). Similarly to the homogeneous and isothermal cases (1 and 3), the negative slope of the source term distribution near the walls for Case 2, Figs. 5(c) and (d), is the consequence of the variation of the optical depth within the enclosure, which is low near the walls and high in the middle. The cells neighbouring the walls emit more energy to the cold surfaces than they receive from the hot regions. As the distance from the walls increases, the medium becomes less influenced by the cold regions, thus leading to a lower absolute value of the source term. At a further distance from the wall, the absolute value of the source term becomes more and more important due to the rapid increase in temperature and concentration, and reaches a very high value at the peak temperature in the middle of the enclosure, where the emitted energy density is significantly higher than the absorbed radiant energy density. The same argument is also applicable to the distribution of the source term of Case 4, Figs. 7(a) and (b), although this distribution is no longer M-shaped. In this latter case, the gas ( $\text{H}_2\text{O}$ ) behaves optically thinner than that in Case 2 ( $\text{CO}_2$ ). However, numerical tests indicate that the distribution

of the source term becomes M-shaped when higher concentrations  $c_0$  (e.g. 0.2) or larger dimensions of the enclosure (e.g. 5 m  $\times$  2.5 m) are used.

The results observed with the mixture case (Fig. 8) are similar to those of Cases 2 and 4, except that there are asymmetrical along  $x$ , Figs. 8(a) and (c), because of the asymmetrical temperature

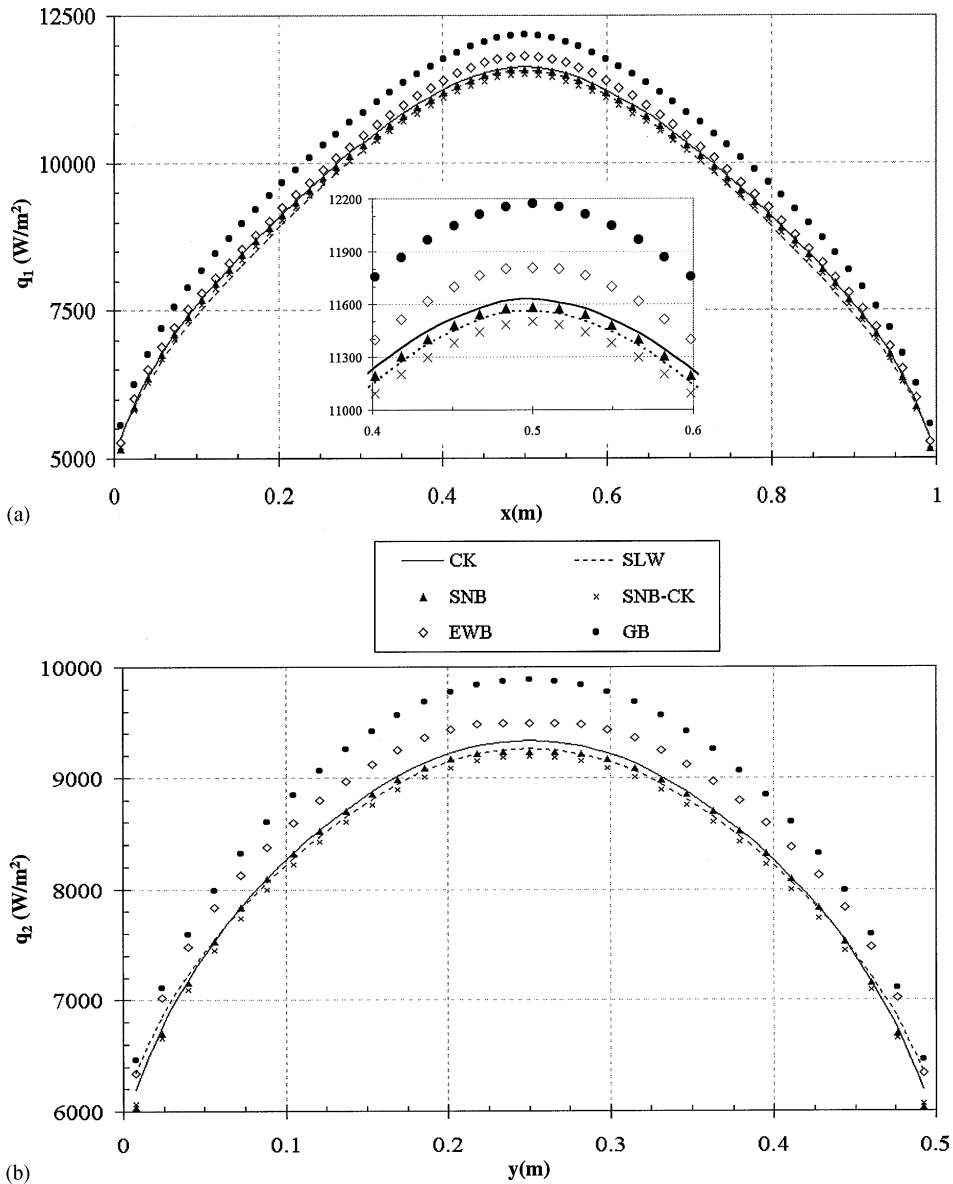


Fig. 5. (a) Evolution of the heat flux  $q_1$  for the non-isothermal and non-homogeneous case with CO<sub>2</sub> (Case 2); (b) Evolution of the heat flux  $q_2$  for the non-isothermal and non-homogeneous case with CO<sub>2</sub> (Case 2); (c) Evolution of the source term  $-(\text{div } q)_x$  for the non-isothermal and non-homogeneous case with CO<sub>2</sub> (Case 2); (d) Evolution of the source term  $-(\text{div } q)_y$  for the non-isothermal and non-homogeneous case with CO<sub>2</sub> (Case 2).

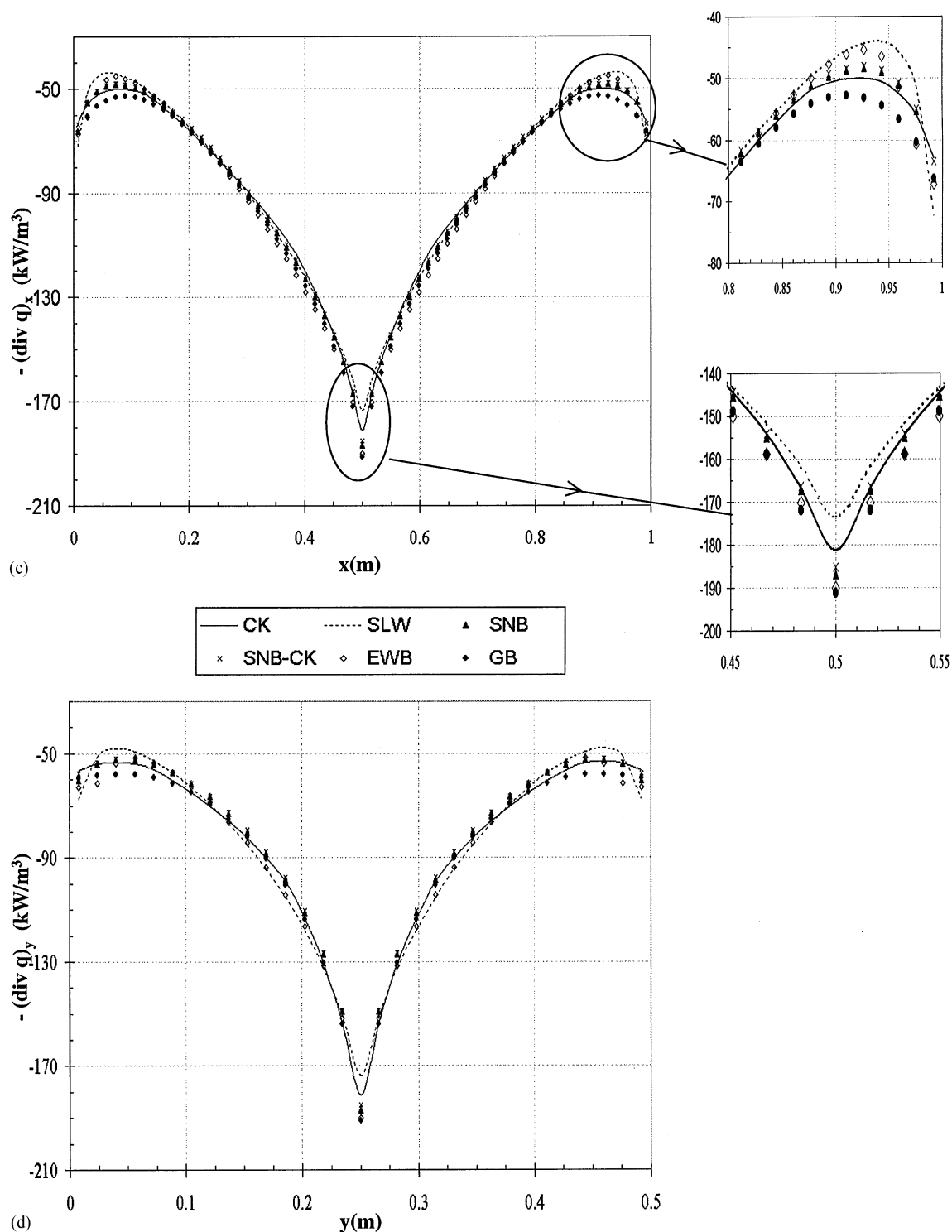


Fig. 5. (Continued).

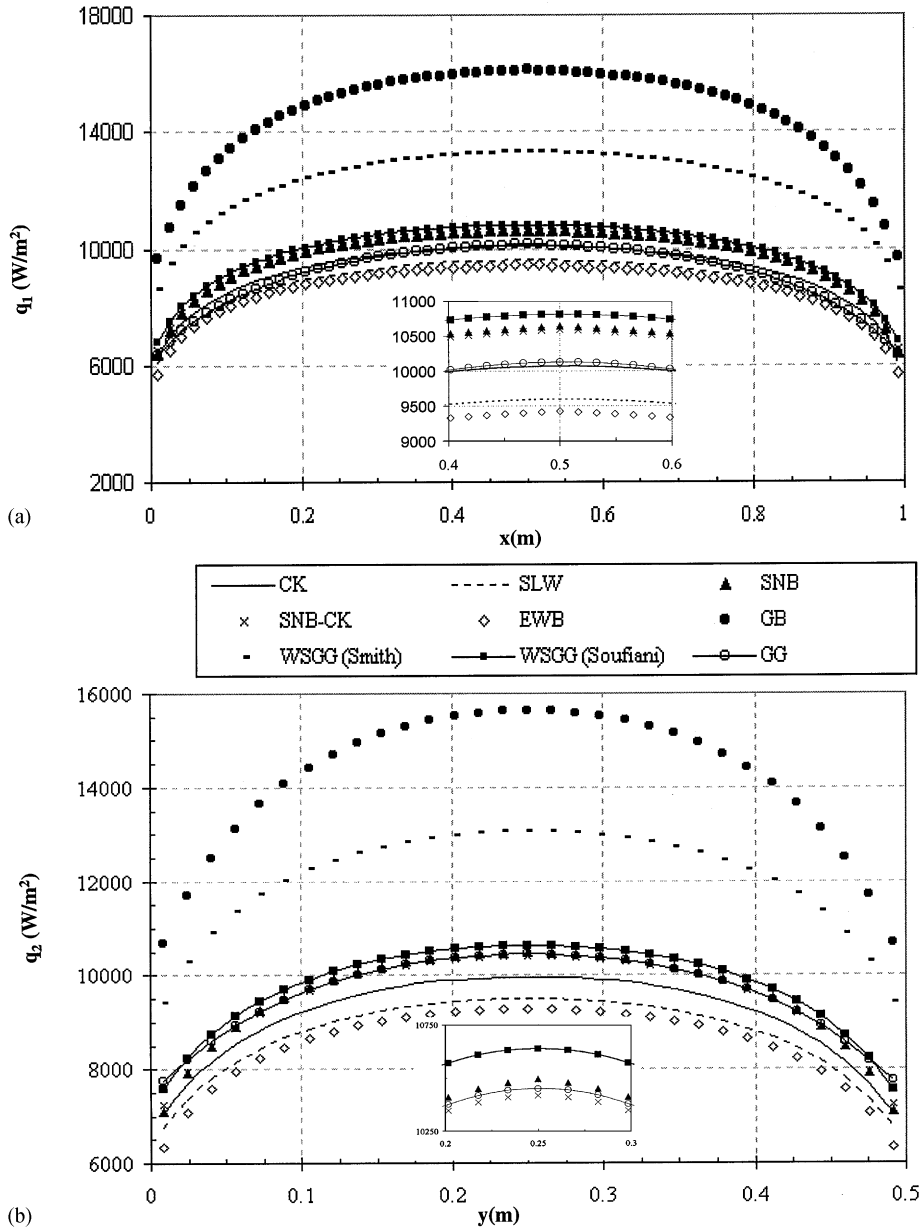


Fig. 6. (a) Evolution of the heat flux  $q_1$  for the isothermal and homogeneous case with H<sub>2</sub>O (Case 3); (b) Evolution of the heat flux  $q_2$  for the isothermal and homogeneous case with H<sub>2</sub>O (Case 3); (c) Evolution of the source term  $-(\text{div } q)_x$  for the isothermal and homogeneous case with H<sub>2</sub>O (Case 3); (d) Evolution of the source term  $-(\text{div } q)_y$  for the isothermal and homogeneous case with H<sub>2</sub>O (Case 3).

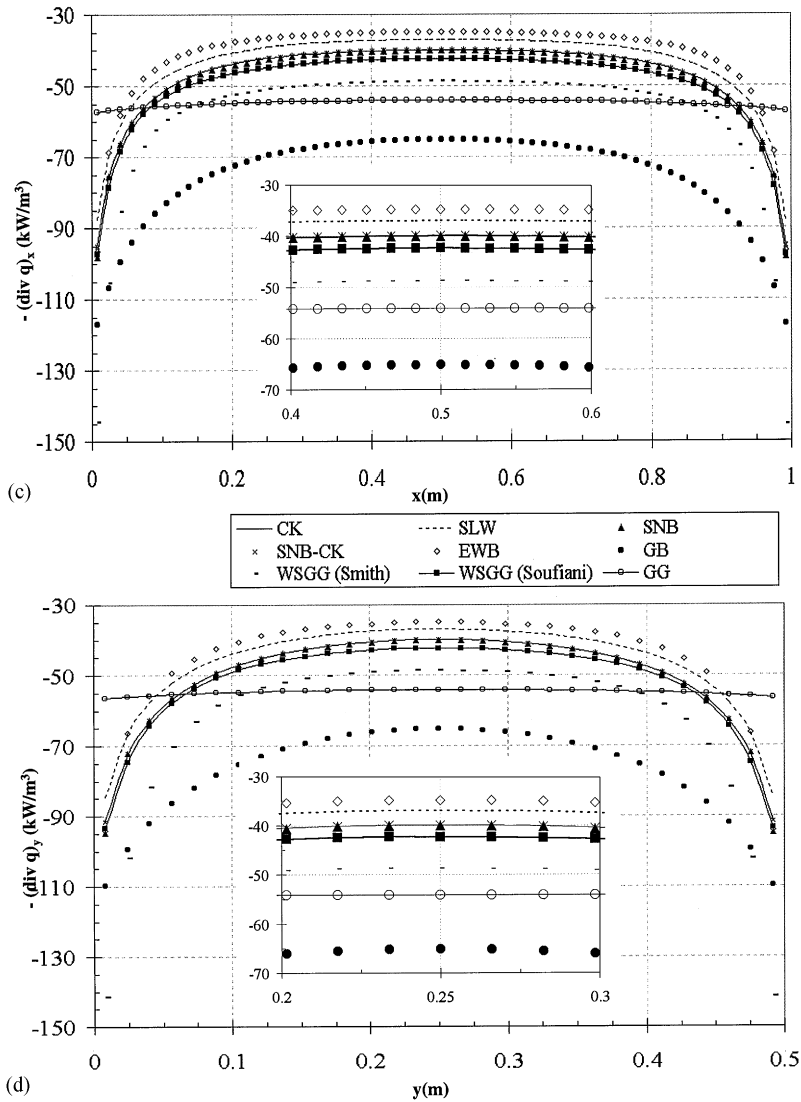


Fig. 6. (Continued).

field. Also, since the temperature gradient at wall 4 is higher than that at wall 2, the heat flux divergence varies more abruptly at wall 4, Fig. 8(c).

### 5.2.2. The reference model

In order to compare the behaviour of each method, it is necessary to define a reference model. This role is often played by the line-by-line method. However, in the present study, we have chosen to use a narrow-band model ( $25 \text{ cm}^{-1}$  bandwidth) as reference. We had the choice between two narrow-band methods, the coefficients of which (mean transmissivities or absorption coefficients) are calculated by allowing for the variation of the absorption coefficient with respect to the wave

number in each band: the SNB or the SNB-CK model. As an excellent agreement between the results of these two methods is observed (Figs. 4–8), the maximum discrepancies being less than 2% for all the cases, either of these two methods could have been chosen. We have taken the SNB as the reference, since this method is more widely accepted (Tables 2–6).

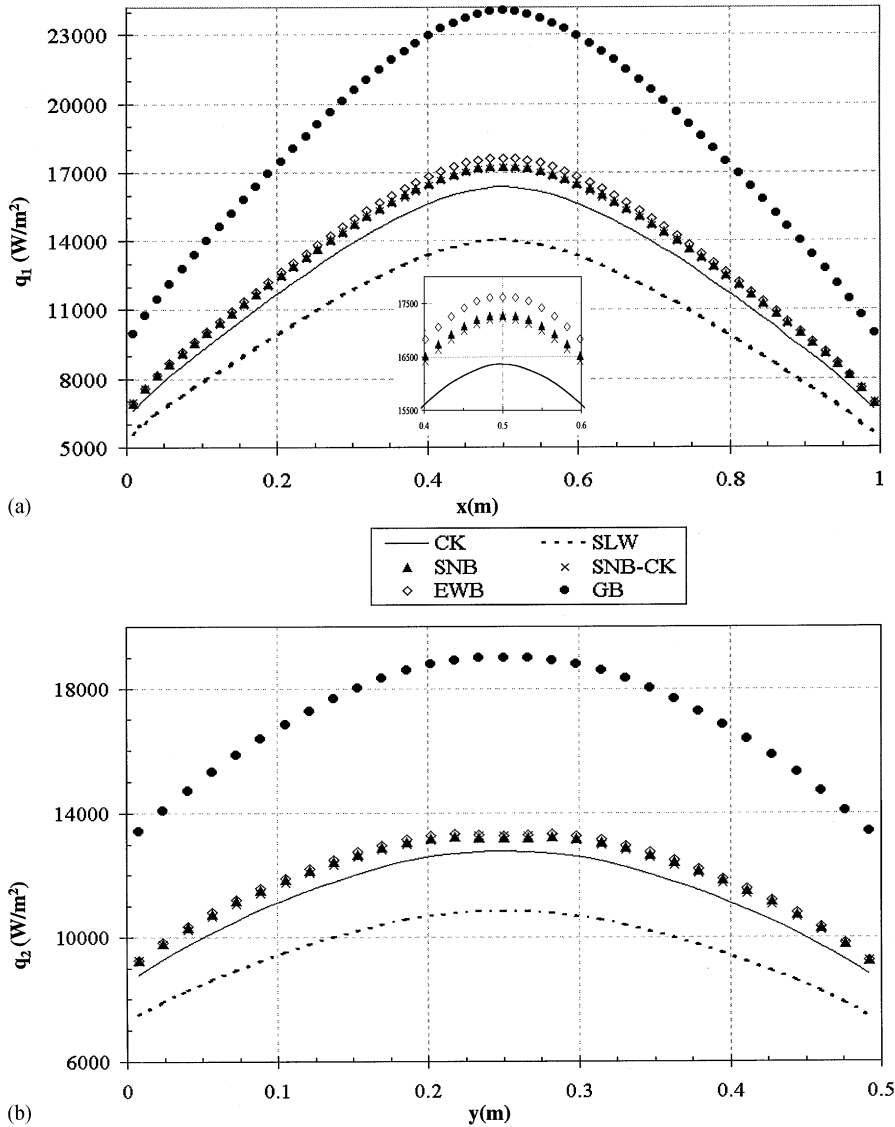


Fig. 7. (a) Evolution of the heat flux  $q_1$  for the non-isothermal and non-homogeneous case with H<sub>2</sub>O (Case 4); (b) Evolution of the heat flux  $q_2$  for the non-isothermal and non-homogeneous case with H<sub>2</sub>O (Case 4); (c) Evolution of the source term  $-(\text{div } q)_x$  for the non-isothermal and non-homogeneous case with H<sub>2</sub>O (Case 4); (d) Evolution of the source term  $-(\text{div } q)_y$  for the non-isothermal and non-homogeneous case with H<sub>2</sub>O (Case 4).

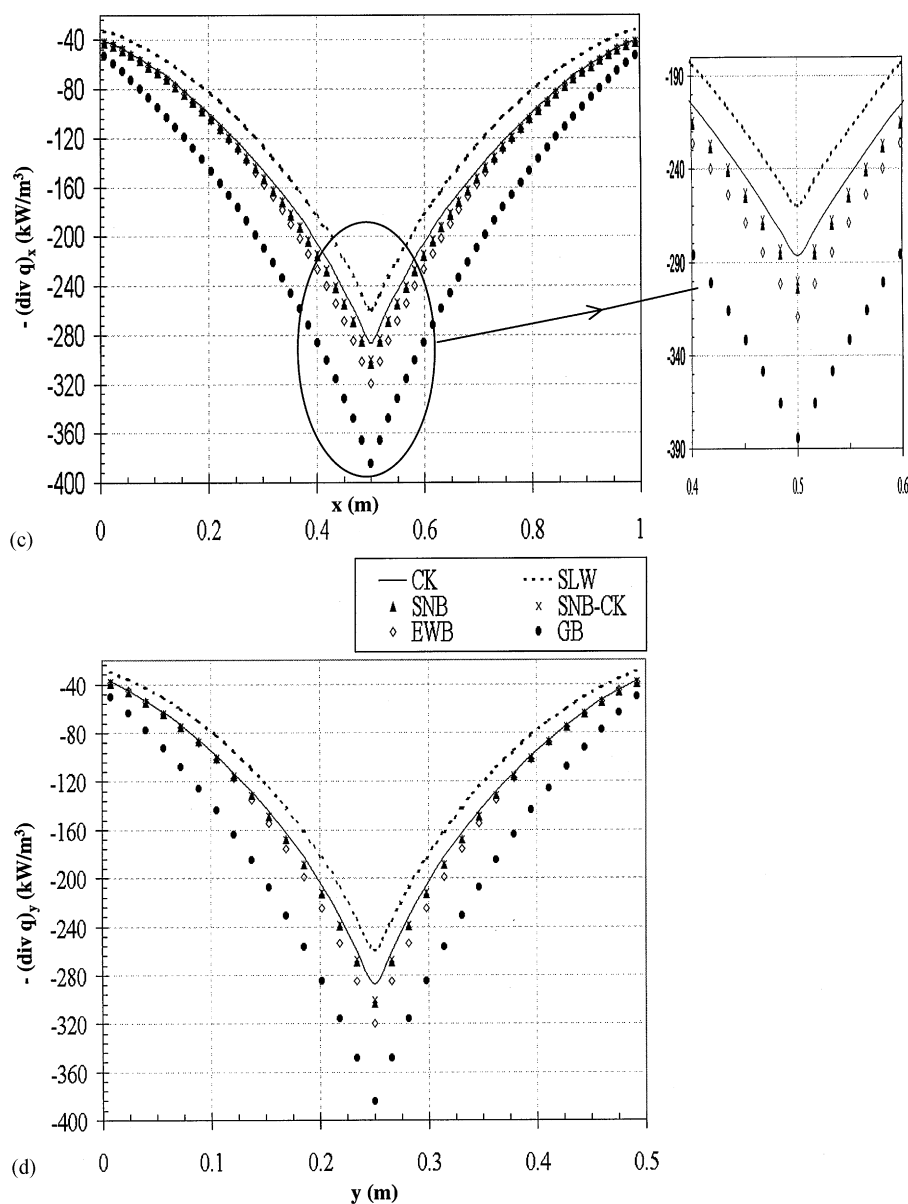


Fig. 7. (Continued).

### 5.2.3. Results of the comparisons

Even though these last two methods give similar results, an important advantage can be given to the SNB-CK model as for the computation time. Since the SNB-CK method is formulated in absorption coefficients, which allows a coupling with a discrete ordinates method, the computation time of this method is reduced by a factor of 6 for CO<sub>2</sub>, 20 for H<sub>2</sub>O and 6 for the mixture case (see Table 7).

The GB model, which is the most rapid (seven times more than the SNB-CK for a single gas) of the three narrow-band models used, cannot claim the same quality of results because of the assumption of a grey gas behaviour in each narrow-band. The discrepancies of the GB method are acceptable for  $\text{CO}_2$ : they vary between 5 and 8.5% for the wall heat fluxes, Figs. 4(a), (b), 5(a), and (b), and between 2.7 and 14.1% for the heat flux divergences, Figs. 4(c), (d), 5(c), and (d). However, the errors of this method become unacceptable for  $\text{H}_2\text{O}$  and the mixture: the discrepancies reach 63% for the divergence in the  $\text{H}_2\text{O}$  homogeneous case, Figs. 6(c) and (d), and 26.6% for the wall heat fluxes in the non-homogeneous case, Figs. 7(c) and (d).

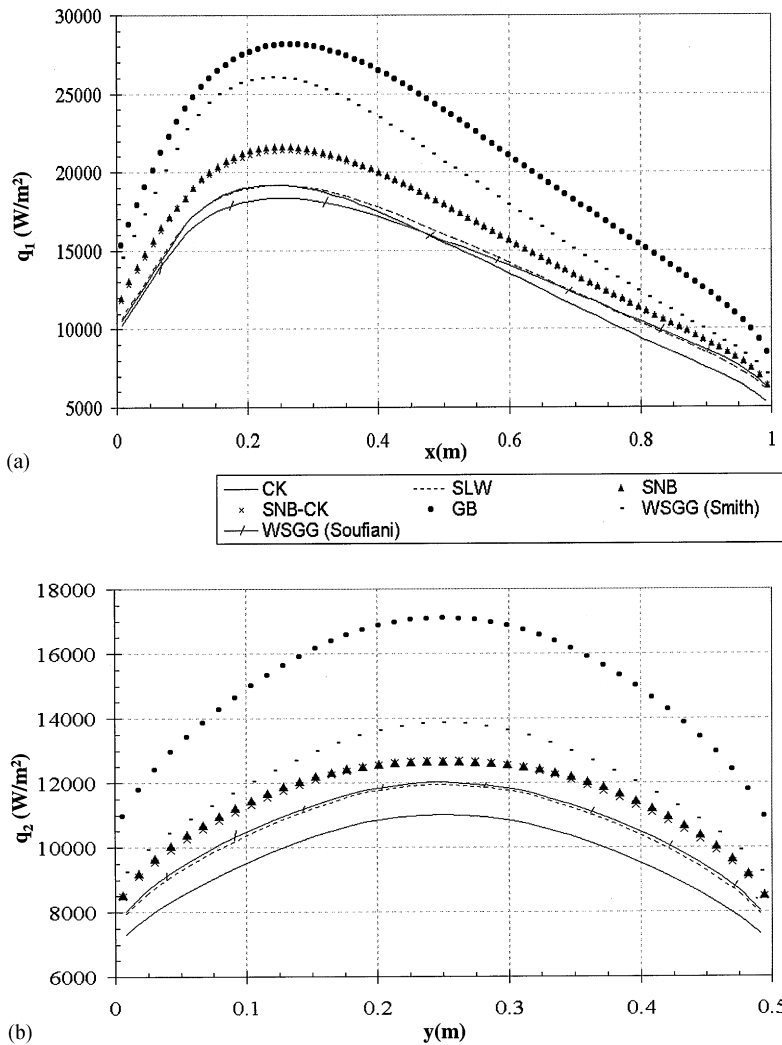


Fig. 8. (a) Evolution of the heat flux  $q_1$  for the mixture case (Case 5); (b) Evolution of the heat flux  $q_2$  for the mixture case (Case 5); (c) Evolution of the source term  $-(\text{div } q)_x$  for the mixture case (Case 5); (d) Evolution of the source term  $-(\text{div } q)_y$  for the mixture case (Case 5).

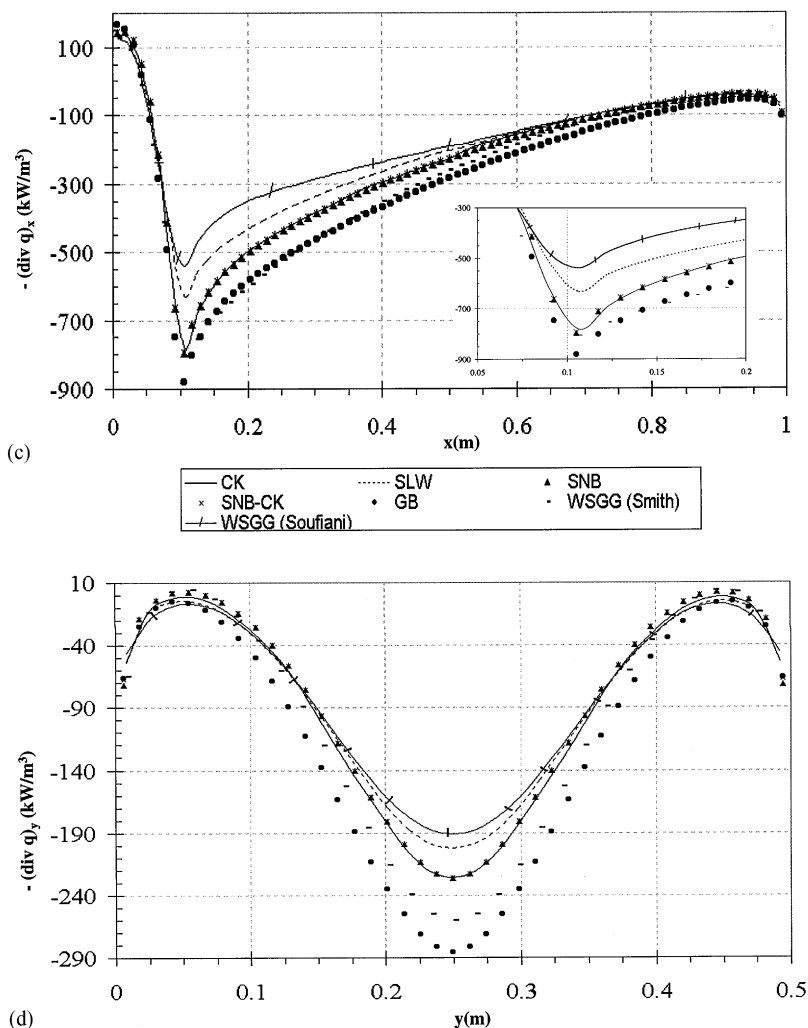


Fig. 8. (Continued).

The optimized CK method, which is about ten times (a factor of 16 for  $\text{CO}_2$  and of 7.5 for  $\text{H}_2\text{O}$ ) more rapid than the SNB-CK for a single participating gas because of its large bandwidths, leads to good results. Regardless of the distributions of concentration and temperature, the relative discrepancies are about 2% for  $\text{CO}_2$  (Figs. 4 and 5) and 5% for  $\text{H}_2\text{O}$  (Figs. 6 and 7). For the mixture case, its computation time is still smaller than that of the SNB-CK method (see Table 7) and the heat flux divergences are accurately calculated, Figs. 8(c) and (d), with discrepancies as low as 1%. However, the performance of this method deteriorates for the wall fluxes: the discrepancies vary between 11 and 13%, Figs. 8(a) and (b).

Results predicted by the EWB model are similar to those of the CK method for the non-isothermal and non-homogeneous cases (2 and 4): the discrepancies are less than 2.5% for the wall heat fluxes, Figs. 5(a), (b), 7(a), and (b), and 5% for the divergences, Figs. 5(c), (d), 7(c), and (d). For

Table 2

Results for the isothermal and homogeneous case with CO<sub>2</sub> (Case 1)

Real-gas model	$q_1$ (W/m <sup>2</sup> )		$q_2$ (W/m <sup>2</sup> )		div $q$ (kW/m <sup>3</sup> )	
	At (0.5, 1.0)	Discrepancies (%)	At (1.0, 0.25)	Discrepancies (%)	At (0.5, 0.25)	Discrepancies (%)
SNB	5537		5479		– 12.59	
SNB-CK	5473	– 1.8	5407	– 1.3	– 12.61	0.2
GB	6009	8.5	5933	8.3	– 14.35	14.1
CK	5415	– 2.2	5371	– 2.0	– 12.52	– 0.6
EWB	5846	5.6	5752	5.0	– 14.54	15.6
SLW	5628	1.6	5565	1.6	– 15.12	20.2
WSGG smith	5760	4.0	5664	3.4	– 15.81	26.3
GG	6000	8.4	6259	14.2	– 32.60	159.8

Table 3

Results for the non-isothermal and non-homogeneous case with CO<sub>2</sub> (Case 2)

Real-gas model	$q_1$ (W/m <sup>2</sup> )		$q_2$ (W/m <sup>2</sup> )		div $q$ (kW/m <sup>3</sup> )	
	At (0.5, 1.0)	Discrepancies (%)	At (1.0, 0.25)	Discrepancies (%)	At (0.5, 0.25)	Discrepancies (%)
SNB	11 581		9230		– 187	
SNB-CK	11 501	– 0.7	9195	– 0.4	– 185	– 1.1
GB	12 174	4.9	9887	7.1	– 192	2.7
CK	11 630	0.4	9337	1.2	– 181	– 3.2
EWB	11 807	2.0	9490	2.5	– 190	1.6
SLW	11 561	– 0.2	9263	0.4	– 173	– 7.5

Table 4

Results for the isothermal and homogeneous case with H<sub>2</sub>O (Case 3)

Real-gas model	$q_1$ (W/m <sup>2</sup> )		$q_2$ (W/m <sup>2</sup> )		div $q$ (kW/m <sup>3</sup> )	
	At (0.5, 1.0)	Discrepancies (%)	At (1.0, 0.25)	Discrepancies (%)	At (0.5, 0.25)	Discrepancies (%)
SNB	10 640		10 495		– 40.0	
SNB-CK	10 585	– 0.5	10 417	– 0.7	– 39.9	~ 0.0
GB	16 101	5.1	15 638	4.9	– 54.1	63.0
CK	10 070	– 5.4	9965	– 5.1	– 39.9	~ 0.0
EWB	9426	– 11.4	9280	– 11.6	– 34.9	– 12.8
SLW	9600	– 9.8	9498	– 9.5	– 37.0	– 7.5
WSGG smith	13 304	25.0	13 062	24.5	– 48.7	21.8
WSGG soufiani	10 808	1.6	10 637	1.4	– 42.3	5.8
GG	10 126	– 4.6	10 449	– 0.4	– 54.1	35.3

Table 5  
Results for the non-isothermal and non-homogeneous case with H<sub>2</sub>O (Case 4)

Real-gas model	$q_1$ (W/m <sup>2</sup> )		$q_2$ (W/m <sup>2</sup> )		div $q$ (kW/m <sup>3</sup> )	
	At (0.5, 1.0)	Discrepancies (%)	At (1.0, 0.25)	Discrepancies (%)	At (0.5, 0.25)	Discrepancies (%)
SNB	17 270		13 205		– 304	
SNB-CK	17 227	– 0.3	13 276	0.5	– 300	– 1.3
GB	24 060	39.3	19 005	43.9	– 385	26.6
CK	16 363	– 5.3	12 785	– 3.1	– 287	– 5.6
EWB	17 613	– 2.0	13 278	0.6	– 319	4.9
SLW	14 031	– 18.8	10 842	– 17.9	– 260	– 14.5

Table 6  
Results for the mixture case (Case 5)

Real-gas model	$q_1$ (W/m <sup>2</sup> )		$q_2$ (W/m <sup>2</sup> )		(div $q$ ) <sub>x</sub> (kW/m <sup>3</sup> )		(div $q$ ) <sub>y</sub> (kW/m <sup>3</sup> )	
	At (0.5, 1.0)	Discr. (%)	At (1.0, 0.25)	Discr. (%)	At (0.24, 0.25)	Discr.(%)	At (0.5, 0.25)	Discr. (%)
SNB	21 630		12 668		– 796		– 226	
SNB-CK	21 373	– 1.2	12 699	0.2	– 782	– 0.5	– 226	~ 0.0
GB	28 142	– 30.1	17 100	35.0	– 792	10.6	– 285	63.0
CK	19 193	– 11.3	11 017	– 13	– 632	– 1.8	– 225	~ 0.0
SLW	19 166	– 11.4	11 944	– 5.7	– 880	– 20.6	– 202	– 7.5
WSGG smith	26 030	20.3	13 868	9.5	– 806	1.3	– 260	21.8
WSGG soufiani	18 330	– 15.3	11 936	– 5.8	– 539	– 32.3	– 190	5.8

Table 7  
Computation time relative to the SNB-CK method

Model	Case 1	Case 2	Case 3	Case 4	Case 5
SNB-CK	1	1	1.1	1.1	9.5
SNB	6	6	22	22	60
GB	1/7	1/7	1/7	1/7	1/4
CK	1/16	1/16	1/7.5	1/7.5	1.2
EWB	9.5	9	9	9	
WSGG	1/500		1/500		1/500
SLW	1/120	1/110	1/120	1/110	1/35

the isothermal cases (1 and 3), although the results are still acceptable, this method is surprisingly less accurate; the discrepancies reach 11.6% with  $\text{H}_2\text{O}$  for the heat fluxes, Fig. 6(b), and 15.6% with  $\text{CO}_2$  for the divergences, Figs. 4(c) and (d). Moreover, since this model is coupled with the ray-tracing method, the computation time is about 9 times that of the SNB-CK method.

As it is usually recognized, the WSGG is indisputably the most rapid model (Table 7) tested in this study, but leads generally to inaccurate results. Two sources of data have been investigated: that of Smith et al. [23], which is the most popular, and that of Soufiani et al. [26]. The discrepancies of the results found when using Smith's data are generally larger: between 15 and 25% (see Figs. 4(c), (d), 6, and 8), except for the  $\text{CO}_2$  heat flux [about 4%, see Figs. 4(a) and (b)]. As for the data of Soufiani et al., they give good results for the homogeneous and isothermal case with  $\text{H}_2\text{O}$  (Fig. 6), but generate greater discrepancies for the mixture case (Fig. 8). The WSGG method seems adequate to obtain a rapid and qualitative description of the radiative transfer in homogeneous enclosures, but cannot yield accurate results.

However, the improvement of the WSGG to the SLW method seems promising. Although the discrepancies for the heat flux divergences are still relatively high (Figs. 4(c), (d), 8(c), (d)), results of the wall heat fluxes for the  $\text{CO}_2$  and mixture cases are acceptable (Figs. 4(a), (b), 5(a), (b), 8(a), (b)). Furthermore, the computation time of this method is significantly shorter than the SNB-CK: about 110–140 times faster for a single gas and 300 times for a mixture. However, if  $\text{H}_2\text{O}$  is the single participating gas, greater discrepancies are observed: about 10% for case 3 and 20% for case 4 (Figs. 6, 7). According to Denison [30], the error comes from their estimation of the hot lines absorption at high temperatures. An upgrading of their data, which would improve the treatment of these hot lines, should lead to better results.

Tables 2–6 summarize the results found for each model for the five cases.

## 6. Conclusions

A comprehensive comparison study is carried out to assess the accuracy and computational efficiency of the most popular real-gas radiation models in a two-dimensional rectangular enclosure. The following conclusions are reached from the results of the present study:

- (1) The statistical narrow-band model (SNB) and the statistical narrow-band correlated- $k$  method (SNB-CK) yield results in very good agreement with each other. Either of them can be used as a benchmark solution in the absence of line-by-line results. These two methods not only predict accurate results for radiation heat transfer calculations, but also yield low-resolution spectral intensities which are required in some other applications. The statistical narrow-band correlated- $k$  method is preferred over the statistic narrow-band model because of its much higher computational efficiency.
- (2) The optimized correlated- $k$  method (CK) leads to accurate and relatively rapid results for single participating gas; however, its efficiency decreases for mixture cases.
- (3) Results of the grey-band method (GB) are qualitatively correct but in serious errors in some cases and therefore this method is not recommended for multi-dimensional radiation heat transfer calculations.

- (4) The exponential wide band model (EWB) yields good results and is a choice for non-grey gas radiation modelling in multi-dimensional problems. However, it should be coupled with the discrete-ordinates method through the methodology of the wide band correlated- $k$  approach in order to gain acceptable computational efficiency.
- (5) The weighted sum of grey gases model (WSGG) yields correct qualitative description of the radiative heat transfer, but cannot lead to accurate results.
- (6) The spectral line-based weighted sum of grey gases model (SLW) is the best choice for multi-dimensional radiation heat transfer calculations based on the considerations of computation time and accuracy. However, better data of hot line absorption of  $\text{H}_2\text{O}$  at high temperatures should be obtained in order to improve the accuracy of this model.

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