# NRC Publications Archive <br> Archives des publications du CNRC 

Lumping errors of analog circuits for heat flow through a homogeneous slab<br>Stephenson, D. G.; Mitalas, G. P.

This publication could be one of several versions: author's original, accepted manuscript or the publisher's version. / La version de cette publication peut être l'une des suivantes: la version prépublication de l'auteur, la version acceptée du manuscrit ou la version de l'éditeur.

## Publisher's version / Version de l'éditeur:

International Developments in Heat Transfer American Society of Mechanical Engineers, Pt. 1, Sect. A, pp. 28-38, 1961-10-01

NRC Publications Archive Record / Notice des Archives des publications du CNRC : https://nrc-publications.canada.ca/eng/view/object/?id=b8123eb2-d6ff-4c15-92db-ca3057df2f92 https://publications-cnrc.canada.ca/fra/voir/objet/?id=b8123eb2-d6ff-4c15-92db-ca3057df2f92

Access and use of this website and the material on it are subject to the Terms and Conditions set forth at https://nrc-publications.canada.ca/eng/copyright
READ THESE TERMS AND CONDITIONS CAREFULLY BEFORE USING THIS WEBSITE.

L'accès à ce site Web et l'utilisation de son contenu sont assujettis aux conditions présentées dans le site https://publications-cnrc.canada.ca/fra/droits
LISEZ CES CONDITIONS ATTENTIVEMENT AVANT D'UTILISER CE SITE WEB.
Questions? Contact the NRC Publications Archive team at
PublicationsArchive-ArchivesPublications@nrc-cnrc.gc.ca. If you wish to email the authors directly, please see the first page of the publication for their contact information.

Vous avez des questions? Nous pouvons vous aider. Pour communiquer directement avec un auteur, consultez la première page de la revue dans laquelle son article a été publié afin de trouver ses coordonnées. Si vous n'arrivez pas à les repérer, communiquez avec nous à PublicationsArchive-ArchivesPublications@nrc-cnrc.gc.ca.

# National Research Council CANADA 

# LUMPING ERRORS OF ANALOG CIRCUITS FOR HEAT FLOW THROUGH A HOMOGENEOUS SLAB 

BY
D. G. STEPHENSON AND G. P. MITALAS

REPRINTED FROM
INTERNATIONAL DEVELOPMENTS IN HEAT TRANSFER american society of mechanical engineers PART I, SECTION A, 1961, P. 28-38

RESEARCH PAPER NO. 139
of the
DIVISION OF BUILDING RESEARCH

This publication is being distributed by the Division of Building Research of the National Research Council. It should not be reproduced in whole or in part, without permission of the original publisher. The Division would be glad to be of assistance in obtaining such permission.

Publications of the Division of Building Research may be obtained by mailing the appropriate remittance, (a Bank, Express, or Post Office Money Order or a cheque made payable at par in Ottawa, to the Receiver General of Canada, credit National Research Council) to the National Research Council, Ottawa. Stamps are not acceptable.

A coupon system has been introduced to make payments for publications relatively simple. Coupons are available in denominations of 5,25 and 50 cents, and may be obtained by making a remittance as indicated above. These coupons may be used for the purchase of all National Research Council publications including specifications of the Canadian Government Specifications Board.


## Lumping Errors of Analog

 Circuits for Heat Flow Through a Homogeneous Slab*ANALYZED

D. G. Stephenson<br>G. P. Mitalas<br>Nalional Research Council<br>Div. of Building Research<br>Orlawa, Canoda


#### Abstract

This paper compares the transfer functions for several electrical analog circuits, both active and passive, with the theoretical functions for heat conduction through a homogeneous slab. The results indicate the magnitude of the errors due to space lumping as a function of non-dimensional frequency for each circuit. This permits the selection of the simplest circuit which will achieve a specified accuracy over a specified frequency range.


## 1. INTRODUCTION

An accurate calculation of the heat flux and temperature distribution through the walls, ceiling or floor of a room requires the simultaneous calculation of the room side surface heat flux of all the elements enclosing the room. Reference [1] discusses the use of an analog computer for this type of building heat transfer problem. It is pointed out in that paper that one of the important considerations in setting up a computer for a room heat transfer calculation is: "How to simulate accurately the wall, roof and floor sections with as few computer elements as possible." This paper presents a method of designing analog circuits to calculate one-dimensional heat conduction through a homogeneous slab with a specified accuracy.

[^0]The analysis differs from earlier studies [2, 3, 4, 5] in two respects: it compares the frequency response of the analog circuits with the theoretical frequency response of a homogeneous slab; and it includes electronic analog circuits with their possibilities for using higher accuracy difference expressions, as well as the passive resistance-capacitance ladder networks.

Previous studies $[2,3,4,5]$ have considered the transient response of passive analog circuits and have concluded that many elements are required if an analog is to represent accurately the response of a slab to a sudden change in the driving function. Only the low frequency components of the outside surface temperature, however, have any significant effect on the conditions inside buildings. Thus any analog that is accurate for frequencies up to the third or fourth harmonic of a diurnal driving function is quite satisfactory for calculating conditions inside a building.

Designing an analog circuit for a limited frequency range leads to simpler circuits than are needed for an accurate response to a step change in the driving function. Thus the frequency response approach is used in this investigation.

Carslaw and Jaeger [6] show that when the temperatures at each surface of a homogeneous slab vary sinusoidally, the surface temperatures and heat flows are related by linear equations which can be expressed as:

$$
\left[\begin{array}{c}
\theta_{\text {out }}  \tag{1}\\
q_{\text {out }}
\end{array}\right]=\left[\begin{array}{cc}
A, & -R B \\
-\frac{D}{R}, & A
\end{array}\right] \cdot\left[\begin{array}{l}
\theta_{\text {in }} \\
q_{\text {in }}
\end{array}\right]
$$

where $\theta=$ temperature
$q=$ heat flux
$A=\operatorname{Cosh}(1+j) \phi$
$B=\frac{\operatorname{Sinh}(1+j) \phi}{(1+j) \phi}$
$D=(1+j) \phi \operatorname{Sinh}(1+j) \phi$
$\phi=\sqrt{\frac{\pi L^{2}}{\alpha P}}$
$R=L / k$
$L=$ thickness of the slab
$k=$ thermal conductivity
$\alpha=$ thermal diffusivity
$P=$ period of the temperature cycle
The square matrix on the right side of (1) is called the transmission matrix for the slab.

Equation (1) can be rearranged to give:

$$
R \cdot\left[\begin{array}{l}
q_{\mathrm{in}}  \tag{2}\\
q_{\mathrm{out}}
\end{array}\right]=\left[\begin{array}{ll}
A / B & ,-1 / B \\
1 / B & ,-A / B
\end{array}\right] \cdot\left[\begin{array}{l}
\theta_{\mathrm{in}} \\
\theta_{\mathrm{out}}
\end{array}\right]
$$

The $D$ term has been eliminated by using the fact that the determinant

$$
\left|\begin{array}{ll}
A & B  \tag{3}\\
D & A
\end{array}\right|=1
$$

If an analog circuit is to calculate accurately the heat flux at the surfaces of a slab it must have transfer functions which are similar to $A$ and $B$ for all frequencies up to the frequency of the highest nonnegligible harmonic of the driving temperatures. In the following sections, the $A$ and $B$ transfer functions are presented for several different analog circuits.

## 2. ANALOG CIRCUITS TO CALCULATE TEMPERATURE AND HEAT FLOW THROUGH A SLAB

The temperature distribution through a homogeneous slab (Fig. 1) is described by
where

$$
\frac{\partial \theta}{\partial t}=\alpha \frac{\partial^{2} \theta}{\partial x^{2}}
$$

$x=$ space coordinate in the direction of heat flow $t=$ time
This partial differential equation can be solved approximately by an analog computer for the temperature at the planes $x=x_{1}, x_{2}-\cdots x_{N-1}$. The method is to replace $\left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{x=x_{i}}$ by a finite difference expression involving the temperatures at the planes $x=x_{0}, x_{1} \cdots x_{N}, x_{0}$ and $x_{N}$ are the two boundaries and the temperatures at these planes are the neces-


FIG. 1 SUBDIVISION OF A SLAB INTO $N$ layers.
sary boundary conditions. The use of a finite difference expression for $\partial^{2} \theta / \partial x^{2}$ converts the partial differential equation into a set of ordinary differential equations which have to be solved simultaneously.

The heat flux through any plane $x=x_{i}$ is given by

$$
\begin{equation*}
q_{i}=-k\left(\frac{\partial \theta}{\partial x}\right)_{x=x_{i}} \tag{4}
\end{equation*}
$$

Thus to evaluate heat flux with an analog computer, it is also necessary to approximate $\frac{\partial \theta}{\partial x}$ by a finite difference expression.

If one neglects the effects of imperfections in the computer components, analog circuits differ in their accuracy only because of the differences in the approximation of the space derivatives on which they are based. In Appendix I the following finite difference expressions for the space derivatives are derived:

$$
\begin{align*}
&-\Delta x\left(\frac{\partial \theta}{\partial x}\right)_{0}=\frac{1+2 a}{a(1+a)} \theta_{0}-\frac{1+a}{a} \theta_{1}+\frac{a}{1+a} \theta_{2} \\
&-\frac{a(1+a)}{6}(\Delta x)^{3}\left(\frac{\partial^{3} \theta}{\partial x^{3}}\right)_{1}-\frac{a(1+a)(1-2 a)}{24} \times \\
&(\Delta x)^{4}\left(\frac{\partial^{4} \theta}{\partial x^{4}}\right)_{1} \tag{5}
\end{align*}
$$

$(\Delta x)^{2}\left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{1}=\frac{2}{a(1+a)} \theta_{0}-\frac{2}{a} \theta_{1}+\frac{2}{1+a} \theta_{2}-\frac{(1-a)}{3} \times$
$(\Delta x)^{3}\left(\frac{\partial^{3} \theta}{\partial x^{3}}\right)_{1}-\frac{\left(1+a^{3}\right)}{12(1+a)}(\Delta x)^{4}\left(\frac{\partial^{4} \theta}{\partial x^{4}}\right)_{1}$

$$
\begin{equation*}
(\Delta x)^{2}\left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{i}=\theta_{i-1}-2 \theta_{i}+\theta_{i+1}-\frac{(\Delta x)^{4}}{12}\left(\frac{\partial^{4} \theta}{\partial x^{4}}\right)_{i} \tag{7}
\end{equation*}
$$

for $1<i<N-1$
Equations (5) and (6) also apply at points $N$ and $N-1$ respectively when $\Delta x$ is replaced by $-\Delta x$ and the subscripts

> 0 by $N$,
> 1 by $N-1$, and
> 2 by $N-2$.
$\Delta x$ is the distance between the equally spaced internal points and

$$
a=\frac{x_{1}-x_{0}}{\Delta x}=\frac{x_{N}-x_{N-1}}{\Delta x} .
$$

The terms including the third and all higher derivatives are usually neglected. This constitutes the error in the finite difference approximations.

This error can be reduced by choosing the value of $a$ so that the coefficients of the neglected derivatives are as small as possible. For instance, when $a=1$, the coefficient of $\frac{\partial^{3} \theta}{\partial x^{3}}$ in (6) is zero so the first neglected term in the expression for $\frac{\partial^{2} \theta}{\partial x^{2}}$ is $\frac{(\Delta x)^{2}}{12}\left(\frac{\partial^{4} \theta}{\partial x^{4}}\right)$. This is the same for all the internal points. The error can be reduced by decreasing $\Delta x$, i.e. increasing the number of lumps. The first two terms which are neglected in the expression for the temperature gradient at the surface, however, are $\frac{(\Delta x)^{2}}{3}\left(\frac{\partial^{3} \theta}{\partial x^{3}}\right)_{1}-\frac{(\Delta x)^{3}}{12}\left(\frac{\partial^{4} \theta}{\partial x^{4}}\right)_{1}$.
When $a=0.5$, the third derivative term is reduced to $\frac{(\Delta x)^{2}}{8}\left(\frac{\partial^{3} \theta}{\partial x^{3}}\right)_{1}$ and the fourth derivative term dis-


FIG. $2 a$ ANALOG CIRCUIT TO COMPUTE SURFACE HEAT FLUX AND INTERNAL TEMPERATURES FOR A HOMOGENEOUS SLAB WHEN SURFACE TEMPERATURE IS GIVEN.


FIG. 26 ANALOG CIRCUIT TO COMPUTE SURFACE TEMPERATURE AND TEMPERATURES THROUGH A HOMOGENEOUS SLAB WHEN SURFACE HEAT FLUX IS GIVEN.
appears. This improvement in the accuracy of the expression for the surface gradient is partially offset by the reappearance of a
$\frac{\Delta x}{6}\left(\frac{\partial^{3} \theta}{\partial x^{3}}\right)_{1}$ in the expression for $\left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{1}$
The circuits for an operational amplifier type of analog shown in Figs. 2a and 2b are based on the difference equations for $a=1.0$
i.e. $-\left(\frac{\partial \theta}{\partial x}\right)_{0}=\frac{3 \theta_{0}-4 \theta_{1}+\theta_{2}}{2 \Delta x}$
$\operatorname{and}\left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{i}=\frac{\theta_{i+1}-2 \theta_{i}+\theta_{i+1}}{(\Delta x)^{2}}$
for $1 \leq i \leq N-1$

$$
\Delta x=L / N
$$

The circuits for $a=0.5$ are only slightly more complicated.

The values of the transmission matrix coefficients for this type of analog circuit have been calculated and the results for $a=1.0$ are given in Table I, and for $a=0.5$ in Table II. The expressions used for calculating these coefficients are derived in Appendix II. It is possible to obtain the matrix coefficients using an analog computer but the results obtained in this way include the errors due to imperfections of the computer components. The results calculated from the expressions in Appendix II represent the performance of an analog with perfect components. The significance of the differences between the matrix coefficients for an analog circuit and the theoretical coefficients for a homogeneous slab are discussed later.

A passive network of resistances and capacitances can also be used to simulate the heat flow in a slab. In this case the temperatures are represented by voltages and the heat flows by currents. Figure 3 is a typical passive analog circuit. The
voltage at any internal point $i$ is described by
$\frac{d}{d t}\left(V_{i}^{\prime}\right)=\frac{1}{R^{\prime} C^{\prime}}\left\{V_{i+1}^{\prime}-2 V_{i}^{\prime}+V_{i+1}^{\prime}\right\}$
Thus if $\frac{1}{R^{\prime} C^{\prime}} \propto \frac{\alpha}{(\Delta x)^{2}}$
this type of circuit will solve the set of ordinary differential equations for the temperatures through a slab.

With this type of analog the surface heat flux is represented by current; thus

$$
\begin{equation*}
q_{0} \propto I_{0}=\frac{V_{0}^{\prime}-V_{1}^{\prime}}{a R^{\prime}} \tag{11}
\end{equation*}
$$

This is equivalent to using

$$
\begin{equation*}
-\left(\frac{\partial \theta}{\partial x}\right)_{0}=\frac{\theta_{0}-\theta_{1}}{a \cdot \Delta x} \tag{12}
\end{equation*}
$$

A Taylor's series expansion about $x=0$ gives

$$
\begin{equation*}
-\left(\frac{\partial \theta}{\partial x}\right)_{0}=\frac{\theta_{0}-\theta_{1}}{a \cdot \Delta x}+\frac{a(\Delta x)}{2!}\left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{0}+\ldots \tag{13}
\end{equation*}
$$

Thus taking $I_{0}$ as proportional to $q_{0}$ involves neglecting $\frac{a \cdot \Delta x}{2}\left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{0}$ and all the higher derivatives. This higher error in the surface heat flux is an important disadvantage of the passive networks compared with the operational amplifier networks.

The $A_{N}$ and $B_{N}$ transmission matrix coefficients have been evaluated for resistance-capacitance networks with $a=0.5$ for $N=3$ (1) 14. (When $a=0.5$ an $N$ lump circuit consists of $N-1$ " T " networks in series.) The results are given in Table III. The


FIG. 3 RESISTANCE-CAPACITANCE CIRCUIT TO SIMULATE HEAT FLOW IN A HOMOGENEOUS SLAB.
method used was similar to the one outlined in Appendix II for the amplifier networks except that the expression for the temperature gradient at the surface was

$$
\begin{aligned}
& \left(\frac{\partial \theta}{\partial x}\right)_{N}=\frac{\theta_{N}-\theta_{N-1}}{a \cdot \Delta x} \\
& \text { where } \Delta x=\frac{L}{\left(\frac{L+2 a-2)}{}\right.}
\end{aligned}
$$

## 3. DISCUSSION OF RESULTS

The heat flows indicated by an analog circuit are given by

$$
\left[\begin{array}{c}
q_{o}  \tag{14}\\
q_{N}
\end{array}\right]=\frac{1}{R}\left[\begin{array}{ccc}
\frac{A_{N}}{B_{N}} & , & \frac{-1}{B_{N}} \\
\frac{1}{B_{N}} & , & \frac{-A_{N}}{B_{N}}
\end{array}\right] \cdot\left[\begin{array}{l}
\theta_{0} \\
\theta_{N}
\end{array}\right]
$$

(The subscripts indicate that the quantity is associated with a lumped analog circuit.) If it is assumed that $\theta_{O}=\theta_{\text {in }}$ and $\theta_{N}=\theta_{\text {out }}$, equation (14) can be subtracted from (2) to give

$$
\left[\begin{array}{l}
q_{\text {in }}-q_{O}  \tag{15}\\
q_{\text {out }}-q_{N}
\end{array}\right]=\frac{1}{R}\left[\begin{array}{l}
\frac{A}{B}-\frac{A_{N}}{B_{N}}, \frac{1}{B_{N}}-\frac{1}{B} \\
\frac{1}{B}-\frac{1}{B_{N}}, \frac{A_{N}}{B_{N}}-\frac{A}{B}
\end{array}\right] \cdot\left[\begin{array}{l}
\theta_{\text {in }} \\
\theta_{\text {out }}
\end{array}\right]
$$

The lefthand column matrix elements are the errors in the heat flux as calculated by the analog circuit.

$$
\text { Let } \begin{align*}
& \frac{A}{B}-\frac{A_{N}}{B_{N}}  \tag{16}\\
&=U_{N} e^{\delta^{\delta}}  \tag{17}\\
& \frac{1}{B}-\frac{1}{B_{N}}  \tag{18}\\
&=V_{N} e^{j^{\delta_{2}}}  \tag{19}\\
& \theta_{\text {in }}=\left|\theta_{\text {in }}\right| e^{j w t} \\
& \theta_{\text {out }}=\left|\theta_{\text {out }}\right| e^{j\left(w t+\delta_{0}\right)}
\end{align*}
$$

The error in the analog heat flow will have its maximum possible value of

$$
\begin{equation*}
\left|q_{\text {in }}-q_{0}\right|_{\max }=\frac{U_{N}\left|\theta_{\text {in }}\right|+V_{N}\left|\theta_{\text {out }}\right|}{R} \tag{20}
\end{equation*}
$$

when $\delta_{0}=\delta_{1}-\delta_{2}+\pi$

The error in the output heat flow is obtained by interchanging $U_{N}$ and $V_{N}$ in (20). Thus, the maximum possible error in the boundary heat flows indicated by the analog can be calculated easily when $U_{N}$ and $V_{N}$ are known.

The results of the matrix coefficient calculations for the operational amplifier and the resistancecapacitance circuits are plotted in Figs. 4a, 4b, and 5 in the form $\frac{A_{N}}{B_{N}}, \frac{1}{A_{N}}$ and $\frac{1}{B_{N}}$. The theoretical values for a homogeneous slab taken from reference 7 are also plotted so the values of $U_{N}$ and $V_{N}$ can be measured directly from the graphs. Values of $U_{N}$ and $V_{N}$ are given in Table IV as functions of $\phi$ and $N$ for the various circuits. These data were measured off large scale graphs similar to Figs. 4 and 5.

The results in Table IV show that the operational amplifier type analog with $a=0.5$ is the most accurate, of the circuits studied, for the computation of surface heat flows.

The following example illustrates how the charts and tables can be used to calculate the number of lumps needed for a particular problem.

## Example 1.

Problem: How many lumps are required in an operational amplifier analog circuit ( $a=0.5$ ) to calculate the heat flow into a 6 -in. concrete slab with an error of not more than $2 \mathrm{Btu} / \mathrm{ft}^{2} \mathrm{hr}$ when the surface temperature has an amplitude of 5 F at a frequency of 4 cycles/day, and the other surface is perfectly insulated.

$$
\begin{aligned}
\alpha & =0.04 \mathrm{ft}^{2} / \mathrm{hr} \\
k & =1.0 \mathrm{Btu} / \mathrm{ft} \mathrm{hr} \mathrm{~F}
\end{aligned}
$$

Solution: $\phi=\sqrt{\frac{\pi L^{2}}{\alpha P}}=\sqrt{\frac{\pi \times 0.25}{0.04 \times 6.0}}=1.8$

$$
R=L / k=0.5 \mathrm{ft}^{2} \mathrm{hr} \mathrm{~F} / \mathrm{Btu}
$$

$$
\text { when } q_{\text {out }}=0
$$

$$
\theta_{\mathrm{out}}=\frac{1}{A} \theta_{\mathrm{in}}
$$

$$
\text { For } \phi=1.8,|1 / A| \doteqdot 0.3
$$

Thus $\left|\frac{\theta_{\text {out }}}{\mid \theta_{\text {in }}}\right| \doteqdot 0.3$
The maximum error in the surface heat flux is

$$
\left|q_{\text {in }}-q_{0}\right|_{\max }=\frac{\left|\theta_{\text {in }}\right|}{R}\left\{U_{N}+V_{N}\left|\frac{\theta_{\text {out }}}{\left|\overline{\theta_{\text {in }}}\right|}\right|\right\}
$$



FIG. 40 POLAR COORDINATE PLOT OF $1 / A_{N}$ AND $1 / B_{N}$ FOR AN OPERATIONAL AMPLIFIER CIRCUIT WITH $a=0.5$ AND $a=1.0$.

Thus if the error is to be less than $2 \mathrm{Btu} / \mathrm{ft}^{2} \mathrm{hr}$

$$
U_{N}+0.3 V_{N}<\frac{R(2)}{\left|\theta_{\mathrm{in}}\right|}=0.2
$$

The following values of $U_{N}$ and $V_{N}$ are taken
from Table IV for $\phi=2.0$. Values for $\phi=1.8$ can be obtained by interpolation but the nearest plotted value of $\phi$ usually will suffice to find the required $N$

$$
\begin{array}{ll}
U_{3}=0.32 & V_{3}=0.04 \\
U_{4}=0.12 & V_{4}=0.01
\end{array}
$$

Thus a 4 -lump circuit meets the requirement.
If the important results of an analog computation are the temperatures at different points through a slab rather than surface heat flux the error in the temperatures may be estimated as follows:
The temperature at plane $x$ in Fig .1 is:

$$
\begin{equation*}
\theta_{x}=\left(A_{x}-\frac{x}{L} \cdot \frac{A B_{x}}{B}\right) \theta_{\mathrm{in}}+\left(\frac{x}{L} \cdot \frac{B_{x}}{B}\right) \theta_{\text {out }} \tag{21}
\end{equation*}
$$

where the matrix elements with subscript $x$ are for $\phi_{x}=x \sqrt{\frac{\pi}{\alpha P}}$ and the quantities without subscripts are for $\phi=L \sqrt{\frac{\pi}{\alpha P}}$. For an analog circuit with $a=1.0$

$$
\begin{equation*}
\theta_{i}=\left(A_{i}-\frac{i}{N} \cdot \frac{A_{N} B_{i}}{B_{N}}\right) \theta_{\mathrm{in}}+\left(\frac{i}{N} \cdot \frac{B_{i}}{B_{N}}\right) \theta_{\text {out }} \tag{22}
\end{equation*}
$$

where $A_{i}$ and $B_{i}$ are matrix elements for an $i$ lump analog circuit with $\phi_{i}=\frac{i}{N} \phi$. For values of $a$, other than unity, the $A_{i}$ and $B_{i}$ values cannot be taken from the tables but can be calculated by the method given in Appendix II. Thus the error in the analog temperature for the plane $x_{i}$ is

$$
\begin{gather*}
\theta_{x}-\theta_{i}=\left\{A_{x}-A_{i}+\frac{x}{L}\left(\frac{A_{N} B_{i}}{B_{N}}-\frac{A B_{x}}{B}\right)\right\} \theta_{\text {in }}+ \\
\frac{x}{L}\left\{\frac{B_{x}}{B}-\frac{B_{i}}{B_{N}}\right\} \theta_{\text {out }} \tag{23}
\end{gather*}
$$

Since these coefficients are functions of $x / L$ as well as $N$ and $\phi$, it seems impractical to prepare tables of the type given for the heat flux errors. The values of the separate factors can be obtained from Table I. The second example illustrates the procedure.

## Example 2.

Problem: Find the maximum possible error in the temperature at $x=0.75 L$ for a $6-\mathrm{in}$. concrete slab which is represented by a four lump operational amplifier analog circuit, $a=1.0$. Properties and boundary conditions same as for example 1.

Solution: $\phi=1.8$ but use 2.0 to avoid need for interpolation of the tables.

$$
\text { Data: } \begin{aligned}
A_{.75 L} & =\frac{1}{.4694} \downharpoonright 85.5 \\
B_{.75 L} & =\frac{1}{.9022} 41.3 \\
A_{3} & =\frac{1}{.4919} \boxed{86.7}
\end{aligned}
$$

$$
\begin{aligned}
B_{3} & =\frac{1}{.9094}\lfloor 50.1 \\
A & =\frac{1}{.2739}\lfloor 115.4 \\
B & =\frac{1}{.7565} \square 68.6 \\
A_{4} & =\frac{1}{.2831} \square 115.9 \\
B_{4} & =\frac{1}{.7526} \square 76.9
\end{aligned}
$$

These give

$$
\left|\theta_{.75 L}-\theta_{3}\right|_{\mathrm{MAX}}=0.01 \mathrm{~F}
$$

$$
\text { and } \quad\left|\theta^{.75 L}\right|=0.78 \mathrm{~F}
$$

Figure 5 shows that, in general, a resistancecapacitance analog circuit requires more lumps than an operational amplifier analog circuic to achieve the same accuracy for heat flow calculations. An


FIG. 4 b POLAR COORDINATE PLOT OF $1 / A_{N}$ and $1 / B_{N}$ FOR A RESISTANCE-CAPACITANCE LADDER NETWORK WITH $a=0.5$.


FIG. 5 POLAR COORDINATE PLOT OF $A_{N} / B_{N}$ FOR A RESISTANCE-CAPACITANCE LADDER NETWORK WITH $a=0.5$ AND AN OPERATIONAL AMPLIFIER CURCUIT WITH $a=0.5$ AND $a=1.0$.
$N$ lump circuit of either type, however, will give exactly the same temperatures at the internal points. It should be possible, therefore, to use a hybrid analog which will have the same accuracy as the operational amplifier type. It would use amplifier adders to calculate the heat fluxes and a less expensive resistance-capacitance network to compute the temperatures.

## ACKNOWLEDGMENTS

The authors gratefully acknowledge the assistance they have received from J. H. Milsum and
D. C. Baxter of the Division of Mechanical Engineering of National Research Council of Canada. This paper is a contribution from the Division of Building Research of National Research Council of Canada and is published with the approval of the Director of the Division.

## REFERENCES

1. Mitalas, G.P., Stephenson, D.G., and Baxter, D.C., 'Use of an analog computer for room air conditioning calculations." Proceedings of the

Second Conference of the Computing and Data Processing Society of Canada, held in Toronto, June 1960, Pp. 175-192 (NRC6099).
2. Paschkis, V. and Heisler, M.P., "The accuracy of measurements in lumped $R-C$ cable circuits as used in the study of transient heat flow," Trans. Amer. Inst. Elect. Engineers, Vol. 63, 1944, p. 165.
3. Klein, E.O.P., Touloukian, Y.S., and Eaton, J.R., 'Limits of accuracy of electrical analog circuits used in the solution of transient heat conduction problems," presented at the Annual Meeting of the Amer. Soc. of Mech. Engineers, 1952 (ASME Paper No. 52-A-65).
4. Lawson, D.I. and McGuire, J.H., '"The solution of transient heat flow problems by analogous electrical networks,' Proce edings, Institution of Mechanical Engineers, (A), Vol. 167, 1953, p. 275.
5. Friedmann, N.E., "The truncation error in a semi-discrete analog of the heat equation," Journal of Math. and Phys, Vol. 35, No. 3, 1956, p. 299.
6. Carslaw, H.S. and J.C. Jaeger, Conduction of heat in solids, Oxford University Press, Second Edition, 1959.
7. Shirtliffe, C.J. and Stephenson, D.G.,

Tables of $\cosh (1+i) X, \frac{\sinh (1+i) X}{(1+i) X}$ and $(1+i) X$
sinh $(1+i) X$. National Research Council, Division of Building Research, Ottawa, January 1961, 71 pp . (NRC6159).

## APPENDIX I: FINITE DIFFERENCE EXPRESSIONS FOR SPACE DERIVATIVES

Let subscript 0 indicate $x=0$, subscript 1 indicate $x=a \cdot \Delta x$, and subscript 2 indicate $x=(1+a) \Delta x$. Roman numeral superscript indicates the order of derivative with respect to $x$.

A Taylor's series expansion about the point $x=a \cdot \Delta x$ gives

$$
\begin{gather*}
\theta_{0}=\theta_{1}-a \cdot \Delta x \theta_{1}^{\mathrm{I}}+\frac{(a \cdot \Delta x)^{2}}{2!} \theta_{1}^{\mathrm{II}}-\frac{(a \cdot \Delta x)^{3}}{3!} \theta_{1}^{\mathrm{III}}+ \\
\frac{(a \cdot \Delta x)^{4}}{4!} \theta_{1}^{\mathrm{IV}}-+\ldots \tag{I.1}
\end{gather*}
$$

and $\theta_{2}=\theta_{1}+\Delta x \theta_{1}{ }^{\mathrm{I}}+{\frac{(\Delta x)^{2}}{2!}}^{2} \theta_{1}{ }^{\text {II }}+{\frac{(\Delta x)^{3}}{3!}}^{3} \theta_{1}^{\text {III }}+$

$$
\begin{equation*}
\frac{(\Delta x}{4!}_{4!}^{\theta_{1}}{ }^{\mathrm{IV}}+\ldots \tag{I.2}
\end{equation*}
$$

Multiplying I. 2 by $a$ and adding to I. 1 gives

$$
\begin{align*}
& \theta_{0}+a \theta_{2}=(1+a) \theta_{1}+\frac{a(1+a)}{2}(\Delta x)^{2} \theta_{1}^{\text {II }}+ \\
& \frac{a\left(1-a^{2}\right)}{6}(\Delta x)^{3} \theta_{1}^{\text {III }}+\frac{a\left(1+a^{3}\right)(\Delta x)^{4}}{24} \theta_{1}^{\text {IV }}+\ldots . \tag{I.3}
\end{align*}
$$

$$
(\Delta x)^{2} \theta_{1}^{\mathrm{II}}=\frac{2}{a(1+a)} \theta_{0}-\frac{2}{a} \theta_{1}+\frac{2}{1+a} \theta_{2}-
$$

$$
\begin{equation*}
\frac{(1-a)}{3}(\Delta x)^{3} \theta_{1}^{\text {III }}-\frac{1+a^{3}}{12(1+a)}(\Delta x)^{4} \theta_{1}^{\text {IV }}-\ldots . \tag{I.4}
\end{equation*}
$$

For the other internal points ( $1<i<N-1$ ) the second derivative expression is similar to (I.4) except that $a=1$, i.e.

$$
\begin{equation*}
(\Delta x)^{2} \theta_{i}^{\mathrm{II}}=\theta_{i-1}-2 \theta_{i}+\theta_{i+1}-\frac{(\Delta x)^{4}}{12} \theta_{i}^{\mathrm{IV}} \tag{I.5}
\end{equation*}
$$

## Differentiating (I.1) gives

$$
\begin{gather*}
\theta_{0}^{\mathrm{I}}=\theta_{1}^{\mathrm{I}}-(a \Delta x) \theta_{1}^{\mathrm{II}}+\frac{(a \cdot \Delta x)^{2}}{2!} \theta_{1}^{\mathrm{III}}- \\
\frac{(a \cdot \Delta x)^{3}}{3!} \theta_{1}^{\mathrm{IV}} \tag{1.6}
\end{gather*}
$$

Equations (I.1) and (I.2) give

$$
\begin{gather*}
\Delta x \theta_{1}^{\mathrm{I}}=-\frac{1}{a(1+a)} \theta_{0}+\frac{1-a}{a} \theta_{1}+\frac{a}{1+a} \theta_{2}- \\
\frac{a}{6}(\Delta x)^{3} \theta_{1}^{\mathrm{III}}-\frac{a(1-a)}{24}(\Delta x)^{4} \theta_{1} \mathrm{IV} \tag{I.7}
\end{gather*}
$$

Multiplying (I.6) by $\Delta x$ and substituting (I.7) and (I.4) for $\Delta x \theta_{1}{ }^{\text {I }}$ and $(\Delta x)^{2} \theta_{1}{ }^{\text {II }}$ respectively gives

$$
\begin{align*}
& (\Delta x) \theta_{0}^{\mathrm{I}}=-\frac{(1+2 a)}{a(1+a)} \theta_{0}+\frac{1+a}{a} \theta_{1}-\frac{a}{1+a} \theta_{2}+ \\
& \frac{a(1+a)}{6}(\Delta x)^{3} \theta_{1}^{\text {III }}+\frac{a(1+a)(1-2 a)}{24}(\Delta x)^{4} \theta_{1}^{\text {IV }} \tag{I.8}
\end{align*}
$$

when $a=1.0$

$$
\begin{equation*}
\theta_{1}^{\mathrm{II}}=\frac{\theta_{0}-2 \theta_{1}+\theta_{2}}{(\Delta x)^{2}}+0 \theta_{1}^{\mathrm{III}}-\frac{(\Delta x)^{2}}{12} \theta_{1}^{\mathrm{IV}} \tag{I.9}
\end{equation*}
$$

$\theta_{0} \mathrm{I}=\frac{-3 \theta_{0}+4 \theta_{1}-\theta_{2}}{2 \Delta x}+\frac{(\Delta x)^{2}}{3} \theta_{1}^{\text {III }}-\frac{(\Delta x)^{3}}{12} \theta_{1}$ IV
when $a=0.5$

$$
\begin{align*}
& \theta_{1}^{\mathrm{II}}=\frac{8 \theta_{0}-12 \theta_{1}+4 \theta_{2}}{3(\Delta x)^{2}}-\frac{\Delta x}{6} \theta_{1}^{\text {III }}-\frac{(\Delta x)^{2}}{16} \theta_{1}^{\mathrm{IV}}  \tag{I.11}\\
& \theta_{0}^{\mathrm{I}}=\frac{-8 \theta_{0}+9 \theta_{1}-\theta_{2}}{3 \Delta x}+\frac{(\Delta x)^{2}}{8} \theta_{1}^{\mathrm{III}}+0\left(\theta_{1}^{\mathrm{IV}}\right) \tag{I.12}
\end{align*}
$$

## APPENDIX II: CALCULATION OF TRANSFER FUNCTIONS FOR AN OPERATIONAL AMPLIFIER TYPE ANALOG CIRCUIT

The sinusoidal temperature and heat flow at the two surfaces of a slab are given by equation (1). A similar expression relates the voltages in an analog circuit which simulates the heat flow and temperature in a slab.

$$
\left[\begin{array}{l}
V_{\theta_{0}} \\
V_{q_{0}}
\end{array}\right]=\left[\begin{array}{ll}
A_{N}, & R B_{N} \\
\frac{D_{N}}{R}, & A_{N}
\end{array}\right] \cdot\left[\begin{array}{l}
V_{\theta_{N}} \\
V_{q_{N}}
\end{array}\right]
$$

The subscript on the $V$ 's indicates the thermal quantity which the voltage represents. For simplicity in notation $\theta_{0}$ is subsequently used in place of $V_{\theta_{0}}$ and similarly for the other quantities with the understanding that it refers to the voltage when used in connection with an analog circuit. $N$ is the total number of lumps that are used in the circuit so $A_{N}$ and $B_{N}$ are the transmission matrix coefficients for an $N$ lump analog circuit. A subscript $i$ on the temperatures and heat flows indicates that the quantity pertains to the output of the $i^{\text {th }}$ lump. Thus the subscripts 0 and $N$ represent the two surfaces.

Case 1. All Lumps the Same. This is the case where $a=1.0$. Equations (5) and (6) become

$$
\begin{gather*}
-\left(\frac{\partial \theta}{\partial x}\right)_{0}=\frac{3 \theta_{0}-4 \theta_{1}+\theta_{2}}{2 \Delta x}  \tag{II.2}\\
\left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{i}=\frac{\theta_{i-1}-2 \theta_{i}+\theta_{i+1}}{(\Delta x)^{2}}  \tag{II.3}\\
\text { for } 1 \leq i \leq N-1
\end{gather*}
$$

and

$$
\begin{equation*}
-\left(\frac{\partial \theta}{\partial x}\right)_{N}=\frac{4 \theta_{N-1}-3 \theta_{N}-\theta_{N-2}}{2 \Delta x} \tag{II.4}
\end{equation*}
$$

$$
\begin{equation*}
\Delta x=L / N \tag{II.5}
\end{equation*}
$$

Hence

$$
\begin{align*}
& q_{N}=\frac{k}{\Delta x}\left(\frac{-3 \theta_{N}+4 \theta_{N-1}-\theta_{N-2}}{2}\right)  \tag{II.6}\\
& \text { or } R q_{N}=\frac{N}{2}\left(-3 \theta_{N}+4 \theta_{N-1}-\theta_{N-2}\right)  \tag{II.7}\\
& \text { and } \frac{d}{d t}\left(\theta_{i}\right)=\frac{a}{(\Delta x)^{2}}\left(\theta_{i-1}-2 \theta_{i}+\theta_{i+1}\right)  \tag{II.8}\\
& \text { for } 1 \leq i \leq N-1 \\
& \text { Expression for } A_{N} .
\end{align*}
$$

When $q_{N}=0 \quad \theta_{0}=A_{N} \theta_{N}$

$$
\begin{equation*}
\text { and } 4 \theta_{N-1}=3 \theta_{N}+\theta_{N-2} \tag{II.9}
\end{equation*}
$$

$$
\text { Let } \begin{aligned}
\frac{\alpha}{(\Delta \mathbf{x})^{2}} & =1 \\
\eta_{i} & =\mathcal{L}\left\{\theta_{i}\right\}=\int_{0}^{\infty} e^{-s t} \cdot \theta_{i} d t
\end{aligned}
$$

Transforming equations (II.8) for $i=N-1$ and (II.9) gives

$$
\begin{equation*}
(s+2) \quad \eta_{N-1}=\eta_{N}+\eta_{N-2} \tag{II.10}
\end{equation*}
$$

$$
\begin{equation*}
4 \eta_{N-1}=3 \eta_{N}+\eta_{N-2} \tag{II.11}
\end{equation*}
$$

Let $s+2=y$

$$
\begin{equation*}
\text { Then } \frac{\eta_{N-1}}{\eta_{N}}=\frac{2}{4-y} \tag{II.12}
\end{equation*}
$$

$$
\begin{equation*}
\frac{\eta_{N-2}}{\eta_{N}}=y\left(\frac{2}{4-y}\right)-1=\frac{3 y-4}{4-y} \tag{II.13}
\end{equation*}
$$

The following general recurrence relationship holds for all other points

$$
\begin{equation*}
\frac{\eta_{N-i}}{\eta_{N}}=y\left(\frac{\eta_{N+1-i}}{\eta_{N}}\right)-\left(\frac{\eta_{N+2-i}}{\eta_{N}}\right) \tag{II.14}
\end{equation*}
$$

for $2 \leq i \leq N$
For each value of $N$ the ratio

$$
\begin{equation*}
\frac{\eta_{0}}{\eta_{N}}=\mathscr{L}\left\{A_{N}\right\} \tag{II.15}
\end{equation*}
$$

To get the steady periodic response of a system to a sinusoidal driving function only requires substituting $j w$ for $s$ everywhere in the expression for the Laplace transform of the response, where $w=2 \pi / P$. Since $a /(\Delta x)^{2}$ has been assumed equal to unity

$$
\begin{equation*}
w=2\left(\frac{\phi}{N}\right)^{2} \tag{II.16}
\end{equation*}
$$

For example: for $N=3$

$$
\begin{align*}
\mathscr{L}\left\{A_{3}\right\} & =y\left(\frac{3 y-4}{4-y}\right)-\frac{2}{4-y} \\
& =\frac{3 y^{2}-4 y-2}{4-y} \tag{II.17}
\end{align*}
$$

Thus $A_{3}=\frac{3 Z^{2}-4 Z-2}{4-Z}$
Where $Z=2+j w=2\left[1+j\left(\frac{\phi}{3}\right)^{2}\right]$
Expression for $B_{N}$.

When $\theta_{N}=0 \quad \theta_{0}=B_{N}\left(R q_{N}\right)$

$$
\begin{equation*}
R q_{N}=\frac{N}{2}\left(4 \theta_{N-1}-\theta_{N-2}\right) \tag{II.20}
\end{equation*}
$$

and

$$
\begin{equation*}
\frac{d}{d t}\left(\theta_{N-1}\right)=\frac{a}{(\Delta x)^{2}}\left(\theta_{N-2}-2 \theta_{N-1}\right) \tag{II.21}
\end{equation*}
$$

The differential equations for the other temperatures are given by II. 8 for $1 \leq i \leq N-2$.

Again let $\alpha / \Delta x)^{2}=1$ and take Laplace transforms of II. 20 and II.21. This gives:

$$
\begin{align*}
\mathscr{L}\left\{R q_{N}\right\} & =\frac{N}{2}\left(4 \eta_{N-1}-\eta_{N-2}\right)  \tag{II.22}\\
\eta_{N-2} & =y \eta_{N-1} \tag{II.23}
\end{align*}
$$

Hence

$$
\begin{equation*}
\mathfrak{\varrho} \frac{\eta_{N-1}}{\left\{R q_{N}\right\}}=\frac{2}{N(4-y)} \tag{II.24}
\end{equation*}
$$

and

$$
\begin{equation*}
\mathfrak{\varrho} \frac{\eta_{N-2}}{\left\{R q_{N}\right\}}=\frac{2 y}{N(4-y)} \tag{II.25}
\end{equation*}
$$

The transform of equation (II.8) gives the general recurrence relationship for

$$
3 \leq i \leq N
$$

$$
\begin{equation*}
\frac{\eta_{N-i}}{\mathcal{L}\left\{R q_{N}\right\}}=y\left(\frac{\eta_{N+1-i}}{\mathscr{L}\left\{R q_{N}\right\}}\right)-\frac{\eta_{N+2-i}}{\mathcal{L}\left\{R q_{N}\right\}} \tag{II.26}
\end{equation*}
$$

For each $N$ the ratio $\eta_{0} / \mathcal{L}\left\{R q_{N}\right\}=\mathcal{L}\left\{B_{N}\right\}$
For example: for $N=3$

$$
\begin{equation*}
\varrho\left\{B_{3}\right\}=\frac{2 y^{2}-2}{3(4-y)} \tag{II.27}
\end{equation*}
$$

The inversion of this is just the same as for the $A_{N}$ terms i.e.

$$
\begin{equation*}
B_{3}=\frac{2 Z^{2}-2}{3(4-Z)} \tag{II.28}
\end{equation*}
$$

where $\quad Z=2\left[1+j\left(\frac{\phi}{3}\right)^{2}\right]$

Case 2. Half Lumps at the Surfaces. This is the situation where $a=0.5$. The finite difference expressions become

$$
\begin{aligned}
& -\left(\frac{\partial \theta}{\partial x}\right)_{N}=\frac{8 \theta_{N}-9 \theta_{N-1}+\theta_{N-2}}{3 \Delta x} \\
& \left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{N-1}=\frac{8 \theta_{N}-12 \theta_{N-1}+4 \theta_{N-2}}{3(\Delta x)^{2}} \\
& \left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{i}=\frac{\theta_{i+1}-2 \theta_{i}+\theta_{i-1}}{(\Delta x)^{2}}
\end{aligned}
$$

$$
\text { for } 1<i<N-1
$$

and

$$
\begin{aligned}
\left(\frac{\partial^{2} \theta}{\partial x^{2}}\right)_{1} & =\frac{8 \theta_{0}-12 \theta_{1}+4 \theta_{2}}{3(\Delta x)^{2}} \\
\Delta x & =\frac{L}{N-1}
\end{aligned}
$$

The development of the expressions for $A_{N}$ and $B_{N}$ for this case is similar to case 1 except that the final step is

$$
\eta_{0}=\frac{3 s+12}{8} \eta_{1}-\frac{1}{2} \eta_{2}
$$

rather than the standard recurrence formula; and $w=2\left(\frac{\phi}{N-1}\right)^{2}$.

## AUTHORS REPRINT*

*Presented at the 1961 International Heat Transfer Conference held August 28-September 1, 1961, University of Colorado, Boulder, Colorado, U.S.A. Papers presented for discussion at this meeting have been published in International Developments in Heat Transfer by The American Soc iety of MechanicalEngineers, 29 West 39th Street, New York 18, N.Y.

TABIE I



| $\varnothing$ | $\#=3$ |  | $y=4$ |  | I $=5$ |  | $\mathrm{H}=6$ |  | Theoretleal |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.5 | . 9003 | - 14.2 | - 9001 | - 14.2 | .9793 | - 14.2 | .9730 | - 14.2 | . 3790 | - 14.2 |
|  | . 9987 | - 5.3 | . 9985 | - 5.4 | . 9305 | - 5.2 | . 3985 | - 5.0 | . 9886 | - 4.8 |
| 1.0 | . 7039 | - 50.2 | . 7761 | - 50.0 | . 7739 | - 49.3 | - 7732 | - 43.9 | . 7731 | - 49.9 |
|  | . 9795 | - 23.1 | . 9713 | - 21.3 | . 9772 | - 20.4 | . 3774 | - 20.0 | . 9765 | - 18.9 |
| 1.5 | . 4919 | - Cu. 7 | . 4755 | - 05.9 | . 4711 | - 35.6 | . 4696 | -85.5 | . 4634 | - 65.5 |
|  | . 9054 | - 50.1 | . 8904 | - 46.4 | . 6974 | - 44.6 | . 3000 | - 43.6 | . 9022 | - 41.3 |
| 2.0 | . 3074 | $-116.9$ | . 2831 | -115.9 | . 2765 | -115.5 | . 2745 | -115.3 | 1.2739 | -115.4 |
|  | .704 | - 82.1 | . 7526 | -76.9 | . 7470 | -74.1 | . 7480 | -72.5 | . 7505 | -63.8 |
| 2.5 | . 2074 | -143.6 | .1761 | -143.0 | . 1676 | -143.5 | . 1640 | $-143.3$ | . 1693 | -143.6 |
|  | . 6456 | -113.9 | . 5040 | -100.7 | . 5725 | -105.2 | . 5700 | -103.1 | . 5215 | - 97.9 |
| 3.0 | . 1499 | -167.0 | . 1133 | -170.6 | . 1040 | -171.1 | . 1005 | -171.1 | . 0993 | -171.9 |
|  | . 5262 | -142.1 | . 4385 | -133.4 | .4177 | $-136.0$ | . 4130 | -133.5 | . 4235 | -126.8 |
| 3.5 | . 1136 | -185.5 | . 0750 | -195.9 | . 0656 | $-13 \mathrm{e} .2$ | . 0613 | $-133.3$ | . 0504 | -200.5 |
|  | . 4313 | -165.7 | . 3260 | -168.1 | . 2932 | -165.8 | . 2300 | -163.4 | . 2991 | -155.6 |
| 4.0 | . 0390 | -202.3 | . 0521 | -216.7 | . 0419 | -224.1 | . 0384 | -225.9 | . 0366 | -223.2 |
|  | . 3561 | -184.8 | . 2425 | -194.0 | . 2110 | -134.3 | . 2012 | -132.7 | . 2072 | -154.2 |
| 5.0 | . 0585 | $-22.50$ | . 0261 | -253.7 | . 0180 | -270.5 | . 0152 | -277.0 | . 0135 | -285.5 |
|  | . 2485 | -211.6 | . 1359 | -236.6 | . 1049 | -245.9 | . 0941 | -240.1 | . 0353 | -241.5 |
| 6.0 | . 0411 | $-237.3$ | . 0141 | -222.2 | . 0022 | -303.4 | . 0062 | -322.3 | . 0050 | $-343.8$ |
|  | . 1796 | -220.3 | . 0780 | -267.5 | . 0525 | -283.3 | . 0436 | -297.7 | . 0421 | -298.8 |
| 7.0 | . 0304 | -245.8 | . 0031 | -300.7 | . 0039 | -337.9 | . 0027 | $-360.8$ | . 0018 | -401.1 |
|  | . 1345 | $-238.9$ | . 0464 | -203.2 | . 0267 | -321.5 | . 0202 | -339.9 | . 0181 | -356.1 |

## TABLE I

## (contincod)

| $\varnothing$ | $1:=3$ | If $=10$ | $\mathrm{N}=15$ | 1 = 20 | Theoretical |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 1.0 | $.7720-49.0$ | . $7728-43.9$ | . $7723-43.9$ | . $7730-47.9$ | $.7731-47.9$ |
|  | . 9777 - 19.5 | . $9780-19.3$ | . $9782-19.1$ | . $9763-19.0$ | . $9785-13.9$ |
| 2.0 | $.2734-115.2$ | . $2732-115.2$ | $.2734-115.3$ | . $2736-115.3$ | . $2739-115.4$ |
|  | . $7503-70.9$ | . $7521-70.2$ | .7543-63.4 | . $7552-69.1$ | .7565-63.2 |
| 3.0 | . $00333-171.2$ | . $0906-171.3$ | . $0308-171.6$ | . $0990-171.7$ | . $0933-171.9$ |
|  | . $4130-130.3$ | . $4161-129.4$ | . $4106-120.0$ | . $4212-127.5$ | . $4235-125.8$ |
| 4.0 | . $0364-227.1$ | . $5860-227.6$ | . $0361-223.3$ | .0353-223.6 | . $0366-223.2$ |
|  | . $1933-163.7$ | . $1997-187.8$ | . $2030-185.3$ | . 2047 -125.1 | -2072-104.? |
| 5.0 | . $0135-231.6$ | . $0131-203.2$ | . $0131-234.7$ | .0132-205.4 | . $0135-285.5$ |
|  | . $0891-247.3$ | . $0394-245.7$ | . $0977-23.5$ | .0931-242.6 | .0953-241.5 |
| 6.0 | . $0050-333.6$ | . $0043-357.4$ | . $0048-310.7$ | . $0540-341.9$ | .0050-343.8 |
|  | . 0357 -302.5 | . $0332-302.4$ | . $0335-300.8$ | . $0404-300.0$ | . $0421-230.3$ |
| 7.0 | . $0019-332.0$ | . 0017 -389.8 | . $0017-335.9$ | . $0017-398.0$ | . $0012-401.1$ |
|  | . $0165-354.2$ | . $0159-357.4$ | . $0164-357.6$ | . $0170-357.2$ | .0101-356.1 |
| 8.0 | . 0007 -426.0 | . 0005 -439.7 | .0005 -450.4 | . $00006-453.7$ | .0007-458.4 |
|  | . $0070-401.3$ | .0065-409.9 | . $0066-413.7$ | .0069-413.6 | . $0076-413.4$ |
| 9.0 | .0003-455.0 | . $0002-486.4$ | . $0002-503.8$ | . $0002-508.8$ | . $0002-515.7$ |
|  | . $0029-443.3$ | . $0020-459.3$ | . $0025-468.8$ | . $0023-470.1$ | . $0031-470.7$ |
| 10.0 | . $0001-499.0$ | . $0001-529.7$ | . $0001-555.9$ | . $0001-563.4$ | $.0001-573.0$ |
|  | $.0013-479.8$ | . $0010-505.0$ | . $0010-522.6$ | . $0011-525.0$ | . $0013-520.0$ |

HoTe The entries in the tables are the quantities shown at right. incoretical is baine as $y=\infty$


## TABEE II




| $\varnothing$ | H $=3$ |  | $2 \mathrm{I}=4$ |  | $1 \mathrm{i}=5$ |  | $N=6$ |  | Theoretical |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.5 | . 9792 | - 14.2 | . 3792 | - 14.2 | . 9754 | - 14.2 | . 9735 | - 14.2 | . 9700 | - 14.2 |
|  | . 3508 | - 4.9 | . 9936 | - 4.8 | . 9355 | - 4.8 | . 9905 | - 4.3 | . 9906 | - 4.8 |
| 1.0 | . 7634 | - 49.5 | . 7634 | - 80.6 | . 7633 | - 49.7 | 1.7707 | - 49.8 | . 7731 | - 49.3 |
|  | . 9505 | - 19.6 | . 9775 | - 10.3 | . 3773 | - 13.9 | . 9775 | - 18.9 | . 0725 | - 10.9 |
| 1.5 | . 4545 | - 84.0 | . 4621 | - 24.5 | . 4639 | - 04.3 | . 4554 | - 05.0 | . 4694 | - 85.5 |
|  | . 9104 | - 42.3 | . 8983 | - 41.2 | . 897 | - 41.1 | . 8381 | - 41.1 | . 9022 | - 41.3 |
| 2.0 | . 2725 | $-111.6$ | . 2661 | -112.8 | . 2675 | $-113.7$ | . 2692 | -114.2 | . 2739 | -115.4 |
|  | . 7721 | - 71.5 | . 7477 | -68.4 | . 7459 | - 66.1 | . 7473 | -63.2 | . 7565 | - 68.8 |
| 2.5 | . 1653 | $-135.0$ | . 1562 | -130.6 | . 2570 | $-140.3$ | . 1505 | -141.4 | .1530 | -143.6 |
|  | .6013 | $-101.2$ | . 5631 | - 95.0 | . 5656 | - 35.4 | . 5677 | - 96.5 | . 5015 | - 97.9 |
| 3.0 | . 1071 | $-157.4$ | . 0923 | -162.9 | . 0927 | $-166.1$ | . 0340 | -168.0 | . 0933 | -171.9 |
|  | . 4470 | $-123.9$ | . 4073 | -124.3 | . 4041 | $-124.0$ | . 4065 | -124.4 | . 1235 | $-126.3$ |
| 3.5 | . 0724 | $-176.1$ | . 0557 | -185.5 | . 0547 | -131.0 | . 0555 | -134.1 | . .0504 | $-200.5$ |
|  | . 3301 | -152.9 | . 2023 | $-150.3$ | . 2785 | $-150.6$ | . 2008 | -151.5 | . 2991 | -155.6 |
| 4.0 | . 0515 | -191.3 | . 0340 | $-205.3$ | . 0322 | -214.6 | . 0326 | -219.3 | . 0356 | -229.2 |
|  | . 2483 | $-172.9$ | . 1334 | -174.5 | . 1074 | -176.2 | . 1091 | -177.9 | . 2072 | $-1 E 4.2$ |
| 5.0 | . 0253 | -215.2 | . 0136 | -241.3 | . 0113 | -25\%.0 | . 0111 | $-250.4$ | . 0135 | $-286.5$ |
|  | . 1519 | $-202.2$ | . 0904 | -216.5 | . 0813 | -223.4 | . 0611 | $-228.0$ | . 0353 | -941.5 |
| 6.0 | . 0297 | $-230.4$ | . 0062 | -26e.2 | . 0042 | $-292.5$ | . 0033 | -303.2 | . 0050 | -343.0 |
|  | . 1025 | -221.1 | . 0445 | -243.1 | . 0347 | $-254.3$ | . 0332 | -273.6 | . 0421 | -293.8 |
| 7.0 | . 0141 | -240.3 | . 0032 | -203.3 | . 0017 | -321.5 | . 0014 | $-344.3$ | . 0018 | -401.1 |
|  | . 0741 | -233.4 | . 0230 | -273.5 | . 0153 | -293.? | . 0134 | -313.9 | . 0161 | -356.1 |

## 2BL: If

## (Continued)

| $\phi$ | $y=0$ |  | $11=10$ |  | $17=15$ |  | $13=20$ |  | Ticorotical |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1.0 | 1.7727 | -49.8 | . 7722 | - 49.2 | . 7727 | -49.8 | . 7729 | - 49.9 | .7731 | - 49.9 |
|  | . 3770 | - 13.9 | . 9780 | - 18.3 | . 9702 | - 18.9 | . 9733 | - 18.9 | . 7785 | - 13.9 |
| 2.0 | . 2711 | $-114.7$ | . 27 ?1 | -115.0 | . 27.31 | -115.2 | . 2735 | $-125.3$ | . 2733 | -215.4 |
|  | . 7503 | - 63.4 | . 75.2 | -68.5 | . 7544 | - 69.7 | . 7553 | - 60.7 | . 7565 | -63.8 |
| 3.0 | . 0360 | $-169.8$ | . 0971 | $-170.6$ | . 0303 | $-171.3$ | . 0383 | $-171.6$ | . 0393 | -171.9 |
|  | . 1119 | -125.2 | . 4155 | $-125.3$ | . 4195 | $-120.3$ | . 4212 | -125.5 | . 4235 | $-126.8$ |
| 4.0 | . 0333 | $-223.3$ | . 0348 | $-225.9$ | . 0350 | -2:7.7 | . 0361 | -223.4 | . 0356 | -223.2 |
|  | .1344 | -130.2 | . 1302 | $-181.5$ | . 2023 | $-102.9$ | . 2046 | -183.5 | . 2072 | -184.2 |
| 5.0 | . 0117 | $-275.0$ | . 0122 | -279.9 | . 0129 | -233.7 | . 0131 | $-234.9$ | . 0135 | -286.5 |
|  | . 0045 | $-233.3$ | . 0075 | $-235.1$ | . 0914 | -233.0 | . 0930 | -240.1 | . 0953 | -241.5 |
| 6.0 | . 0040 | -324.3 | . 0042 | -332.1 | . 0046 | $-333.9$ | . 0047 | -341.1 | . 0050 | -343.3 |
|  | . 0346 | -284.0 | . 0365 | $-203.2$ | . 0332 | -294.4 | . 0403 | $-296.3$ | . 0421 | -293.3 |
| 7.0 | . 0013 | -370.1 | . 0014 | $-302.1$ | . 0015 | $-393.2$ | . 0017 | $-395.3$ | . 0018 | -401.1 |
|  | . 0136 | $-331.6$ | . 0145 | -340.4 | . 0161 | -343.1 | . 0169 | -352.1 | . 0181 | -350.1 |
| 0.0 | . 0004 | -411.3 | . 0005 | -429.4 | . 0005 | -846.4 | . 0006 | -451.9 | . 0007 | -453.4 |
|  | . 0052 | $-375.6$ | . 0056 | -309.2 | . 0054 | -802.0 | . 0369 | -407. 4 | . 0076 | -413.4 |
| 9.0 | . 0002 | -448.2 | . 0002 | -473.7 | . 0202 | $-492.4$ | . 0002 | $-506.3$ | . 0002 | -515.7 |
|  | . 0020 | -415.4 | . 0021 | $-435.3$ | .0025 | -455.3 | . $00<7$ | -462.1 | .0031 | -470.7 |
| 10.0 | . 0001 | -400.0 | . 0001 | $-514.7$ | .0001 | $-580.9$ | . 0001 | -560.0 | . 0001 | -573.0 |
|  | . 0003 | -451.0 | . 0008 | $-476.3$ | . 0003 | -506.5 | . 0011 | $-516.1$ | . 0013 | -528.0 |

HOTE The entries in the tables are the quantities shom at rieht. Theoretical is same as $\quad \pi=\infty$


2AUR： $1 T x$



| $\varnothing$ | 183 3 |  | $\mathrm{i}=4$ |  | 11 $=5$ |  | $n=3$ |  | 2ncoreticos |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $0 . j$ | －$\therefore 7 \%$ | － 24.1 | ． 678 | －16．2 | ． 72 | －14．2 | － | － 14.2 |  |  |
|  |  |  |  |  |  |  |  |  |  |  |
|  | －57 | － 5.4 | － 342 | － 5.0 | －3034 | － 2.9 | －－3） | －$\because 6$ | ． 0.0 | － $0 \cdot 3$ |
| 1.0 | ． 752 | －AB． | ． 767 | － | .776 | － 40. | ．757 | －$\because$ | ． 77.2 | － 49.9 |
|  | 0.637 | － 21.2 | ．972） | －1s．9 | ． 57.9 | － 15.5 | ． 072 | － 1 | － 005 | － 13.9 |
| 1.5 | ． 39 | －-17 | －くら？ | －8．5 | －435 | － 3.4 | ． 4.46 | －$\square_{0}$ | ． 464 | －5．5 |
|  | ． 30 | － 5.1 | ．6754 | － 4.3 | ．67\％ | － 8 So | －6？ | －$\because 2$ | ． 222 | － 51.3 |
| 2.0 | ． 2635 | $-104$. | ．230 | －113． | ．2．3 | $-112$. | ． 29 | －11．7 | ． 270 | $-113.4$ |
|  | － 225 | －71．${ }^{5}$ | ． 35 | －\％． 7 | ．7245 | $-7.8$ | ． 7.3 | －$\because$－ | ．75．5 | － 6.0 |
| 2.5 | ．182 | －121． | ．145 | －13． | ．1）2 |  | ．1， 1 | －1\％ | ． 24 | $-14.6$ |
|  | － 228 | － 95.8 | ． 300 | －So．？ | $\cdot>$ | － | ． 0312 |  | ．501\％ | － 07.5 |
| $\because 6$ | ． 07.0 | －1） 5 | －620 | －1\％0． | －Wu1 | －120．1 | －915 | －1． 0 | －0．0． | －171．9 |
|  | ． 230 | －11\％．6 | ． 322 | $-12.7$ | －＜n | $-2250$ | －\％ | －12t | ．223 | －12 ． 3 |
| 3.5 | ． 454 | －14．4 | － 4 | $-17 \%$ ？ | －30， | －139．1 | .022 | －15． | ． 04 | $-200.5$ |
|  | ． 127 | －120． | ． 2111 | －14． | ． 2.17 | －131． | ．23 | －13．0 | ． 25.1 | $-13 \% .6$ |
| 4.0 | .927 | $-152.7$ | －27 | －1030 | ． 2.7 | －297． | ． 030 | －21\％． | ．026 | $-22.2$ |
|  | ． 105 | －130．4 | ． 103 | －15． | ．15：4 | －175． | ．1，2 | －17．3 | .2072 | $-194.2$ |
| 5.0 | ． 122 ， | －12．0 | $.00,2$ | －21： 5 | ．00y2 | $-24.2$ | －9\％边 | －2． 1 | .015 | $-230.5$ |
|  | ． 0432 | $-15.2$ | ． 0.92 | －155．7 | ． $3: 77$ | －217．2 | $00^{3}$ | －29． | ．935． | $-2 \times 1.5$ |
| $\because$ | $\cdot 001$ | －17．4 | －On， 4 | －223．0 | －00 | －270． | $\cdots 2$ | $-295.3$ | ．05：9 | $-4.03$ |
|  | ． 9280 | －151．1 | ． 312 | －21\％．6 | ． 2111 | $-2 \%$ | ． 224 | －2 |  | －20． |
| 7.0 | ． 023 | －170．7 | ． 14 | －25－3 | .6011 | －23．7 | － 1 c | －2． | ． 10 | －01．1 |
|  | ． 111 | $-1 / 6.1$ | ． $0 \times 2$ | $-22.2$ | － 7 | $-29.2$ | －$\quad 7$ | －． | ．111 | －35．1 |

IADL: TII
(Contimued)

| $\varnothing$ | $\mathrm{N}=8$ |  | ii $=10$ |  | $\pi=12$ |  | $y=14$ |  | Toosetical |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1.0 | .771 | -290 | . 7720 | - 40.0 | . 7728 | - | . 7727 | - | .771 | -49. |
|  | . 577 | - 15.1 | . 773 | -19.1 | - 7 | - 15.3 | . 3762 | - 15.0 |  | - 13.0 |
| 2.0 | . $270 \%$ | -114. | . 2717 | -11ヶ.9 | . 2725 | -115.1 | . 272 | -i15.2 | . 27 | -115.6. |
|  | .75, 1 | -50.2 | .7502 | - 2.1 | .7>22 | - 6 | . 7534 | - 0 | . 755 | - 50. |
| 3.6 | . 0051 | $-16.2$ | -Cof. 7 | -17\% | .0375 | -170.6 | 0.00 | -171.2 | .05\% | $-171.6$ |
|  | . 4043 | -12. 7 | . 1115 | -126.3 | . 4155 | -126. | .4176 | -125. | . 52 | -130 |
| 4.0 | . 0331 | $-222.5$ | .0.44 | -225.2 | .0551 | -2? | .035 | -297. | . 300 | -223. |
|  | .1052 | -122.2 | .15id | $-103.1$ | .1909 | $-15 \pm .5$ | . 20 | -1. 1.7 | . 2072 | -1\%4.2 |
| 5.0 | . 0112 | $-27.1$ | .0120 | -270.7 | .124 | -21 | .912\% | -2.2.3 | . 13 | $-23!$ |
|  | .0778 | $-235$ | . 241 | -2,0.0 | - 97 | -29 |  | -2.3.9 | . 955 | -2ci. 5 |
| 6.0 | .007 | -320.1 | 061 | - 30. | $\cdots \mathrm{Ca}$ | -84.7 | . 2085 | -7. 7 | - 050 | -4\% |
|  | .0,03 | -255.2 | .0 .41 | -2, 1.2 | . 0.35 | -29. 0 | .672 | -25.4 | . 2421 | $-2.2 .6$ |
| 7.6 | . 012 | -352. 6 | .9014 | -75.7 | .015 | -. 0.3 | .11: | -5\% | . 2010 | $-42.1$ |
|  | . 0111 | -3, 1.1 | .011 | -\%2.1 | . 145 | -87.2 | .15: | - , ¢ | . 101 | -2,50.1 |
| 0.0 | . 0004 | -680. | .000: | -424. | -905 | -iy. 2 | . 0305 | $-4^{2}$ | . 2007 | -480. |
|  | .0330 | -972.2 | - 3043 | - 290.1 | .3055 |  | - 06 | -43.2 | $\cdots 9$ | -41..4 |
| 9.0 | .0901 | $-433.4$ | . 0001 | -45.4 | .0002 | -43. | .602 | -6, 2 | -0\%? | -515.7 |
|  | . 0014 | -603. 2 | . 0017 | -4.30.7 | .0020 | -ta7. 5 | . 0025 | -459. 3 | . 9.1 | -470.7 |
| $10.0$ | . 0000 | $-4.1 .3$ | .0900 | $-5040$ | .0rol | -52. | .0301 | -5<1.7 | . 0001 | -57. |
|  | -1) 05 | -439.1 | - 6000 | -475.7 | . 0037 | -454.5 | - noon | -2.0 | . 031 | $-523.1$ |

HoT: 'ine entrios in tho twoles are the cuantities siona at rifht, zeoretion is save es it $=\infty$



| ormiatzomal ampinthi ailajob |  |  |  |  |  |  |  |  |  |  |  |  Rerwim |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $a=0.5$ |  |  |  |  |  | $a=1.0$ |  |  |  |  |  | $a=0.5$ |  |  |  |  |  |
| f103 | $\mathrm{r}=4$ | 1*25 | Nam 6 | $\mathrm{y}=3$ | $N=10$ | $\mathrm{I}=3$ | 1! 1 \% | श= | $3 \times$ | 2=6 | $x=30$ | İm | I= 4 | こiこ5 | 3 F 5 | N: $=3$ | $x=20$ |
| . 012 | . 00 | . 00 | . 00 | . 00 | .00 .00 | . 03 | $\begin{array}{r} .01 \\ .00 \end{array}$ | . 00 | $.08$ | . 00 | . 00 | $\begin{array}{r} .02 \\ .01 \end{array}$ | .00 | . 00 | .00 | .00 | . 00 |
| . 017 | . 01 | .00 .00 | . .00 | . 00 | .00 | . 097 | . 05 | . 03 | . 01 | . 00 | . 0101 | - 0 | . 05 | . 01 | . 00 | . 00 | .00 |
| .14 <br> .02 | .03 | . 08 | . 02 | .00 | $.00$ | .25 .13 | . 15 | . 1125 | $\begin{aligned} & .03 \\ & .03 \end{aligned}$ | . 22 | . 02 | -30 | . 14 | .03 | . 05 | . 00 | $.00$ |
| $.32$ | $.12$ | . 05 | . 01 | $.01$ | $.00$ | $.57$ | $\begin{array}{r} .10 \\ .10 \\ \hline \end{array}$ | . 25 | $.29$ | $.11$ | . $\quad 2$ | $\begin{aligned} & .55 \\ & .13 \end{aligned}$ | . 35 | . 15 | . 15 | . 06 | $.04$ |
| $\begin{array}{r}.71 \\ .04 \\ \hline\end{array}$ | $\begin{array}{r}.27 \\ .02 \\ \hline\end{array}$ | . 13 | .05 | 0.02 | .00 | $\left\lvert\, \begin{aligned} & 2.03 \\ & .27 \end{aligned}\right.$ | . 69 | $\begin{array}{r}.46 \\ .07 \\ \hline\end{array}$ | . 30 | . 20 | . 15 | 1.20 <br> .16 | -6) | . 36 | $.22$ | . 13 | $.07$ |
| $\begin{array}{r} .25 \\ .03 \\ .03 \end{array}$ | $\begin{array}{r} .52 \\ .05 \\ \hline \end{array}$ | $.23$ | $\begin{aligned} & .13 \\ & .03 \\ & \hline \end{aligned}$ | $\begin{array}{r} .04 \\ .02 \\ \hline \end{array}$ | . 01 | $\begin{array}{\|} 2.55 \\ \hline .16 \\ \hline \end{array}$ | $\begin{array}{r} 1.05 \\ .10 \\ \hline \end{array}$ | $\begin{aligned} & .76 \\ & .07 \end{aligned}$ | .83 <br> .05 | - ${ }^{5}$ | . 25 | 1.00 <br> .17 | 1.00 .09 | . 60 | . 04 | . 15 | $.12$ |
|  | $\begin{array}{r}.06 \\ .03 \\ \hline\end{array}$ | .42 | $\begin{array}{r} .23 \\ .03 \\ \hline \end{array}$ | $\begin{aligned} & .10 \\ & .62 \end{aligned}$ | $.05$ |  | 2.55 .05 | 1.12 <br> .04 <br> 1.25 | $\begin{aligned} & .05 \\ & .05 \\ & \hline \end{aligned}$ | $\begin{array}{r} .52 \\ \hline \end{array}$ | $\begin{array}{r} .32 \\ .02 \end{array}$ |  | $\begin{array}{r}1.45 \\ .10 \\ \hline\end{array}$ | . 89 | . 904 | -02 | -15 |
|  | 1.30 .04 | $\begin{gathered} .67 \\ .0 \% \end{gathered}$ | . 39 | .14 <br> .02 | .03 |  | $\begin{array}{r}2.07 \\ .05 \\ \hline\end{array}$ | 1.55 .04 | 1.2 .03 | . 75 | . 52 |  | 2.04 .09 | 1.32 .06 | . 28 | . 47 | $.28$ |
|  | $\begin{array}{r} 2.45 \\ .04 \\ \hline \end{array}$ | $\begin{array}{r} 1.41 \\ .03 \end{array}$ | $\begin{array}{\|} .66 \\ .05 \\ \hline \end{array}$ | $\begin{array}{r} .34 \\ .02 \end{array}$ | $\begin{array}{r} .10 \\ .01 \\ \hline \end{array}$ |  | $\begin{array}{r}3.23 \\ .04 \\ \hline\end{array}$ | 2.5 <br> .01 | 2.05 .03 | 1.35 .01 | . 04 |  | $\begin{array}{r}3.26 \\ .07 \\ \hline\end{array}$ | $\begin{array}{r}2.20 \\ .06 \\ \hline\end{array}$ | $\begin{array}{r}1.62 \\ .03 \\ \hline\end{array}$ | . 07 | . 54 |

na o top number in each corapartwont is $U_{i 1}$ ana botiom one $V_{i 1}$


[^0]:    *Tables I -IV have been deposited as Document 6679 with the American Documentation Institute Publications Project. A copy may be secured by citing the Document number and by remitting $\$ 1.25$ for photoprints, or $\$ 1.25$ for 35 mm microfilm.

